



제 CG-2014-1029 호

안 전 인 증 서

(사업장명) 키토코리아

(소재지) 경기 안성시 서운면 동촌리 안성제4 일반산업단지 E블럭 6-1

위 사업장에서 제조하는 아래의 품목이 「산업안전보건법」 제34조 및 같은 법 시행규칙 제58조의4제4항에 따른 안전인증 심사결과 안전·보건기준에 적합하므로 안전인증표시의 사용을 인증합니다.

_____ 품 목 :	호이스트	_____
_____ 형식(용량):	KD-ER2-015(1.5 ton)	_____
_____ 인증번호 :	14-CG2AC-1029	_____
_____ 인증기준 :	위험기계·기구 의무안전인증기준 (고용노동부고시 제2012-33호)	_____
_____ 인증조건 :	산업안전보건법 "제34조 준수"	_____

2014년 06월 10일

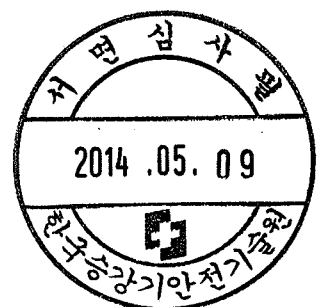
한국승강기안전기술원 이사장



【별지 제4호서식】

동 일 형 식 일 람 표

사업장명	KITO CORP.	개정일자 및 번호	2014.04.29	인증번호	
형식 및 모델번호		동일형식 항목 및 내역			비고
형식번호	모델번호	동일형식 항목1	동일형식 항목2	동일형식 항목3	
KD-ER2-015	KITO-ER2D015S-S	Lift max 30m 권상모타 1.8kW .S : 5.4m/min .IS: 5.3/0.9m/min Inverter control	횡행모터 0.4kW .S : 24m/min .L: 12m/min .IS:24/4m/min .IL:12/2m/min	전기Trolley 결합 type	
	KITO-ER2D015S-L				
	KITO-ER2D015S-IS				
	KITO-ER2D015S-IL				
	KITO-ER2D015IS-S				
	KITO-ER2D015IS-L				
	KITO-ER2D015IS-IS				
	KITO-ER2D015IS-IL				
	KITO-C-ER2D015S-S				
	KITO-C-ER2D015S-L				
	KITO-C-ER2D015S-IS				
	KITO-C-ER2D015S-IL				
	KITO-C-ER2D015IS-S				
	KITO-C-ER2D015IS-L				
	KITO-C-ER2D015IS-IS				
KITO-C-ER2D015IS-IL					
				전기Trolley 결합 Clean type	



Handwritten mark resembling a stylized '4' or a signature.



안 전 인 증 서

정호엔지니어링

경기도 광명시 노온사동 440-5

위 사업장에서 제조하는 아래의 품목이 산업안전보건법 제34조 및 같은 법 시행규칙 제58조의4제4항에 따른 안전인증 심사 결과 안전·보건기준에 적합하므로 안전인증표시의 사용을 인증합니다.

품 목

양중기용 과부하방지장치

형식·모델/용량·등급/인증번호

형식·모델	용량·등급	인증번호
JDL-100	J-2	12-AV2BJ-0009

인 증 기 준

방호장치 의무안전인증 고시(고용노동부고시 제2010-36호)

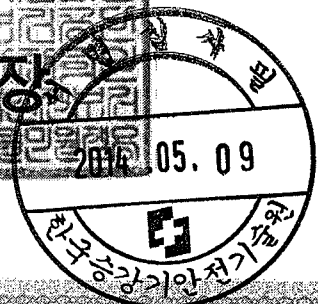
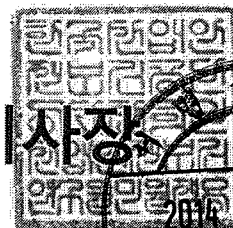
인 증 조 건

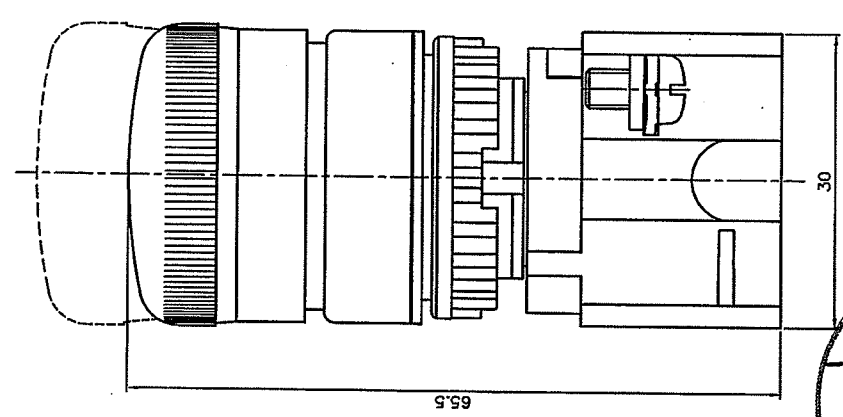
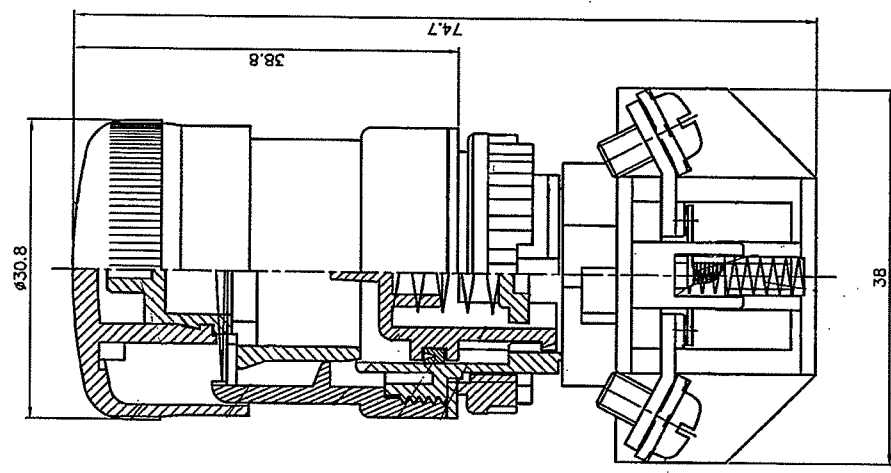
아래 주소에서 생산되는 제품에 한함.

정호엔지니어링, 경기도 광명시 노온사동 440-5

2012년 06월 11일

한국산업안전보건공단 이사장

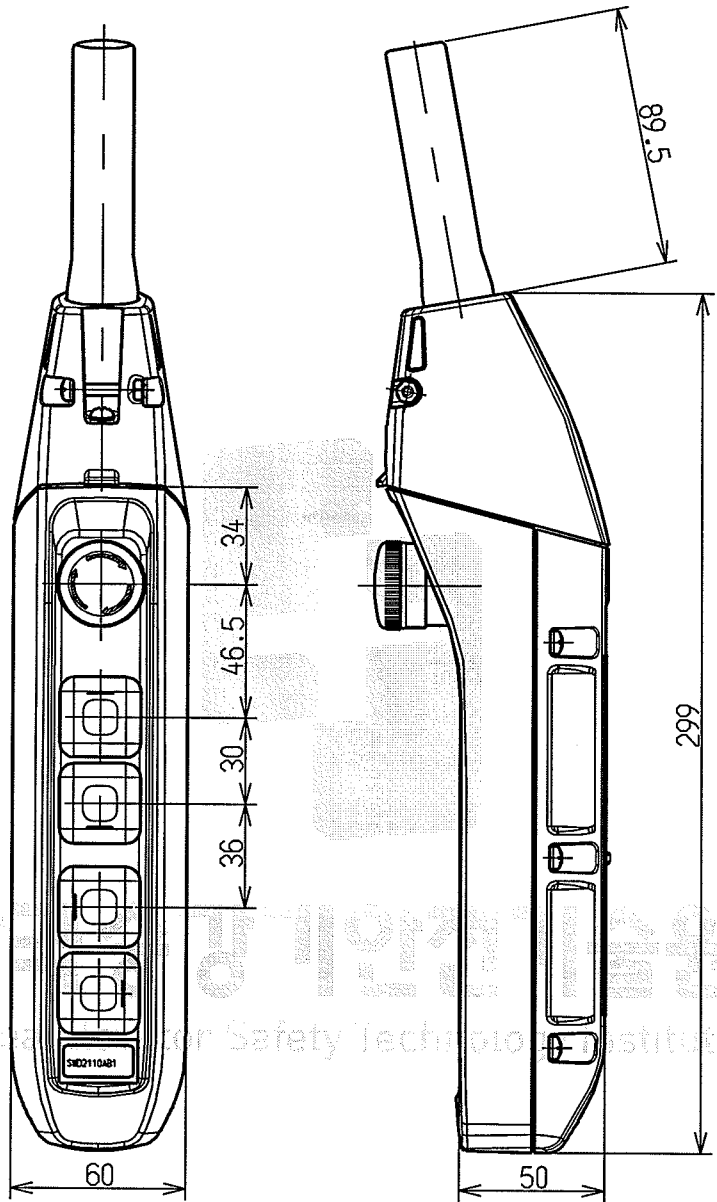




2014.05.05
 한국과학기술진흥원
 天得科技股份有限公司
 TEND TECHNOLOGY CO., LTD.

圖號	T2-BKH	單位	mm	材質	表面處理	圖名	T2 BKH 連鎖開關
出例	2:1	投影法	第一角	顏色	顏色	品名	T2 BKH 連鎖開關
檢具承認	設計課 95.05.24 吳宗達	檢具處理	研裝部 95.05.24 周啟祥	核對	研裝部 95.05.24 鍾偉誌	品准	品檢部 95.05.24 林建宏
檢具承認	研裝部 95.05.24 周啟祥	核對	研裝部 95.05.24 鍾偉誌	品准	品檢部 95.05.24 林建宏	品保	品保部 95.05.24 林建宏
檢具承認	研裝部 95.05.24 周啟祥	核對	研裝部 95.05.24 鍾偉誌	品准	品檢部 95.05.24 林建宏	品保	品保部 95.05.24 林建宏
檢具承認	研裝部 95.05.24 周啟祥	核對	研裝部 95.05.24 鍾偉誌	品准	品檢部 95.05.24 林建宏	品保	品保部 95.05.24 林建宏
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Revision	Incidence	Description	Date	Charge	Approved

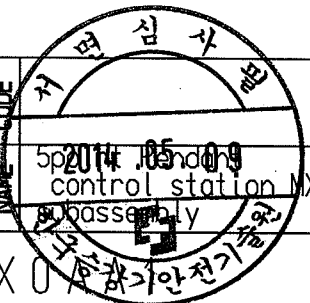


The lifting and lowering push buttons are marked with \updownarrow for single speed or \updownarrow for dual speed.
 The traveling push buttons are marked with E W or N S depending on the installed direction.

⑥
⑤
④
③
②
①

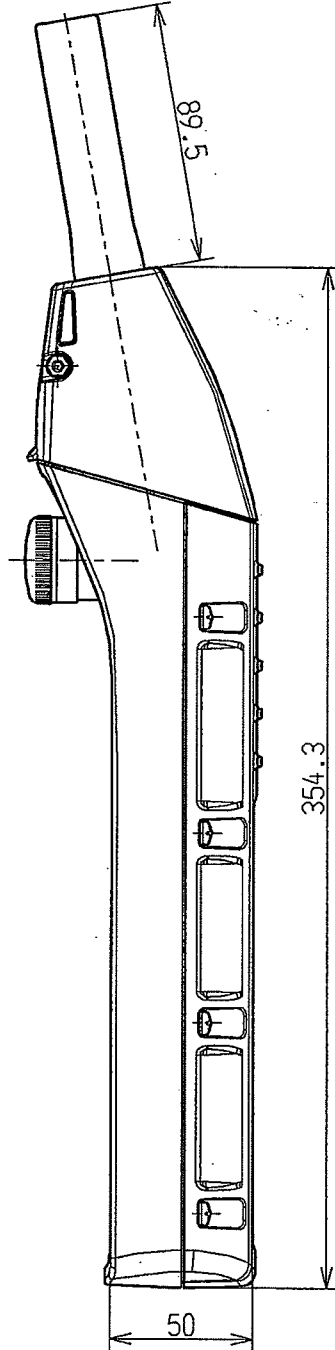
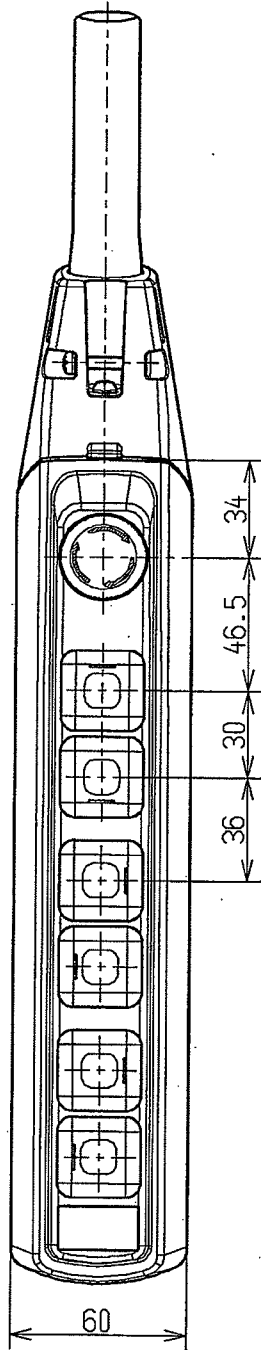
NOTE

APPROVED	H.FURIYA	CHECKED	T.HATANO	DESIGNED	KOBAYASHI	DRAWN	KOBAYASHI	SCALE	-	DWG. NO.	SWD2XX07	NAME CODE	2014.05.09 control station MXX assembly
Date issued	09.04.21		09.04.21		09.04.21		09.04.21						



Revision	Incidence	Description	Date	Charge	Approved

E
W
S
N



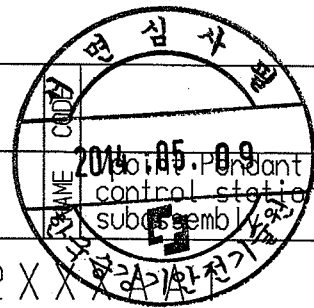
⑥
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NOTE

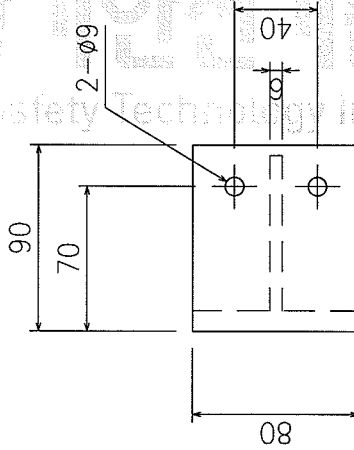
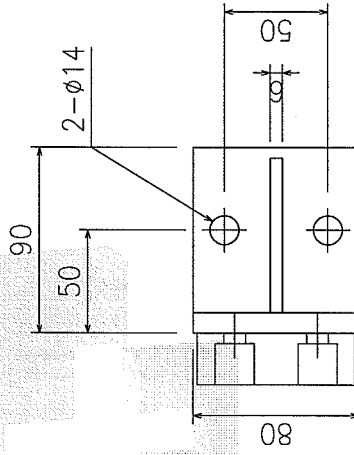
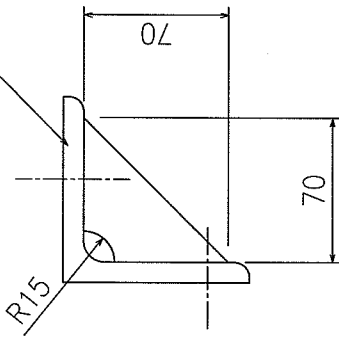
APPROVED	ISHIKAWA	CHECKED	FURIYA	DESIGNED	KOBAYASHI	DRAWN	KOBAYASHI	SCALE	-
Date issued	08.02.08		08.02.08		08.02.08		08.02.08		

DWG. NO., NOS./UNIT MATERIAL

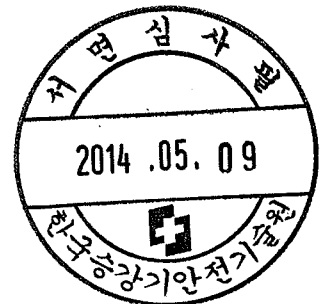
SWD2XXX



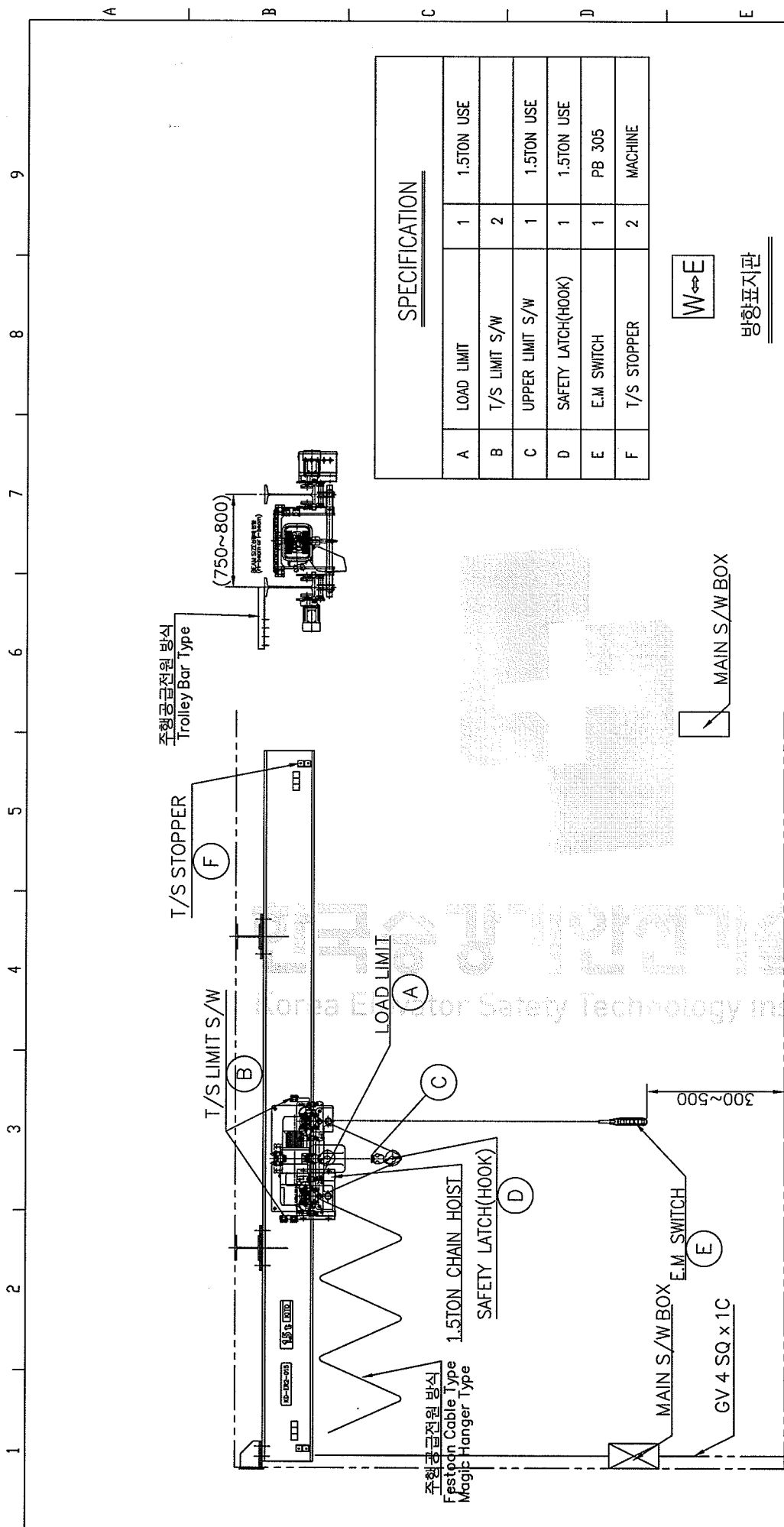
L - 90x10t



한국승강기안전기술원
Korea Elevator Safety Technology Institute



No	Part Name	Description	Mat'l	Unit	Q'ty	REVISION	Remark
	TITLE		SS400		4	4	CRS/SH
STOPPER - traversing							
STOPPER							
(주)KITO KOREA							
APPROVED J.S. CHO J.S. CHO W.H.LEUN W.H.LEUN							
DRAWN							
DATE							
CONTENTS							
REV.							
SCALE							
REV.							



SPECIFICATION

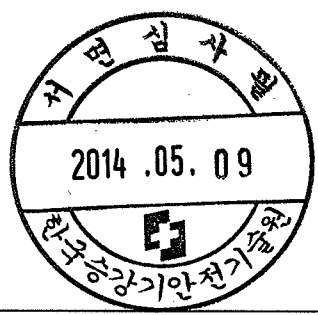
A	LOAD LIMIT	1	1.5TON USE
B	T/S LIMIT S/W	2	
C	UPPER LIMIT S/W	1	1.5TON USE
D	SAFETY LATCH(HOOK)	1	1.5TON USE
E	E.M SWITCH	1	PB 305
F	T/S STOPPER	2	MACHINE

W ↔ E

방향표지판

NOTE

1. 점검설비는 현장여건에 맞추어 설치한다.
2. 사용설명서를 제공한다.



형식번호 : KD-ER2-015

<p>1.5T MONO RAIL HOIST SAFETY DRAWING</p>					
<p>제출번호 DRAWING NO.</p>	<p>제출일자 DATE</p>	<p>작성 DRAWN</p>	<p>검토 CHECKED</p>	<p>설계 DESIGNED</p>	<p>명칭 TITLE</p>
		EUNWON HEE	EUNWON HEE	EUNWON HEE	EUNWON HEE
		J.S. CHO	J.S. CHO	J.S. CHO	J.S. CHO
<p>제출회사 DRAWN APPROVED</p>		<p>주식회사 KORON CORP</p>			
<p>제출일자 DATE</p>		<p>제출인 DRAWN</p>			
<p>제출인 REV.</p>		<p>제출인 CITY</p>			
<p>CONTENTS</p>					
<p>MONO RAIL O</p>					
<p>제출인 REV.</p>		<p>제출인 SCALE</p>		<p>제출인 REV.</p>	

형식번호 : KD-ER2-015

Model number.

KITO-ER2D015S-S

KITO-ER2D015S-L

KITO-ER2D015S-IS

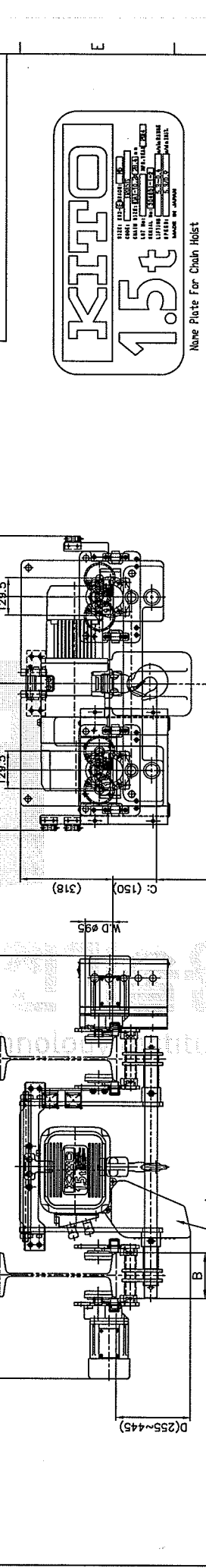
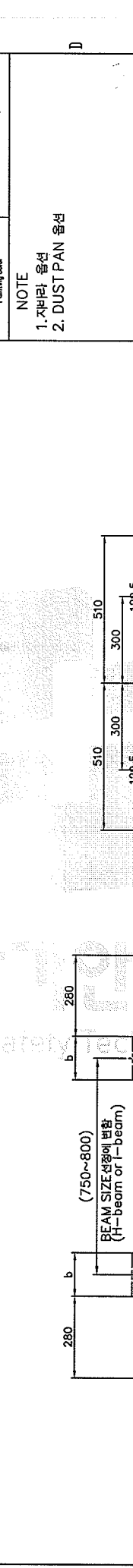
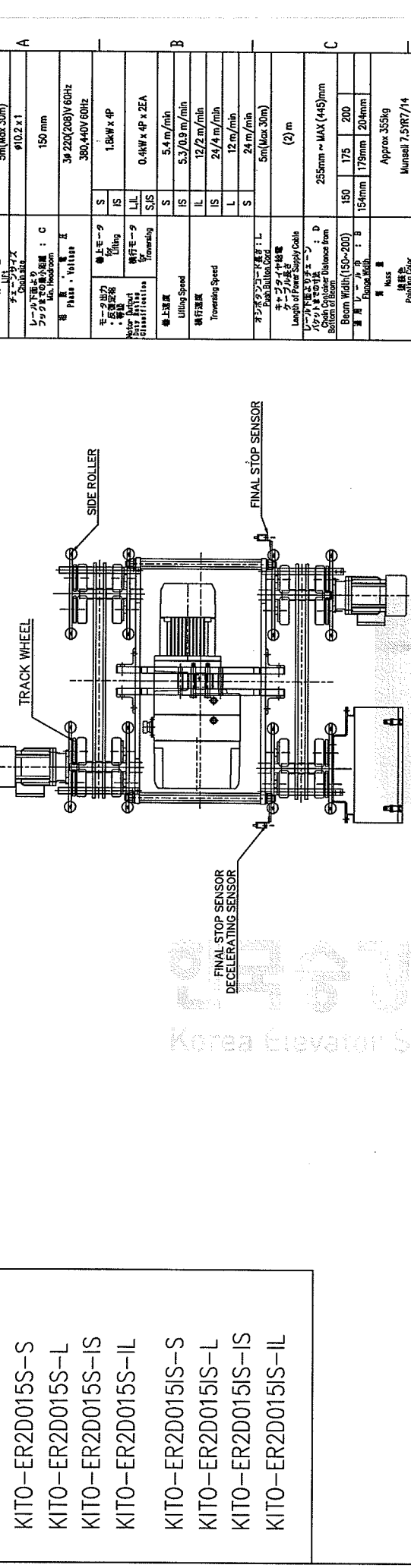
KITO-ER2D015S-IL

KITO-ER2D015S-S

KITO-ER2D015S-L

KITO-ER2D015S-IS

KITO-ER2D015S-IL



None Plate for Chain Hoist

None Plate for Trolley

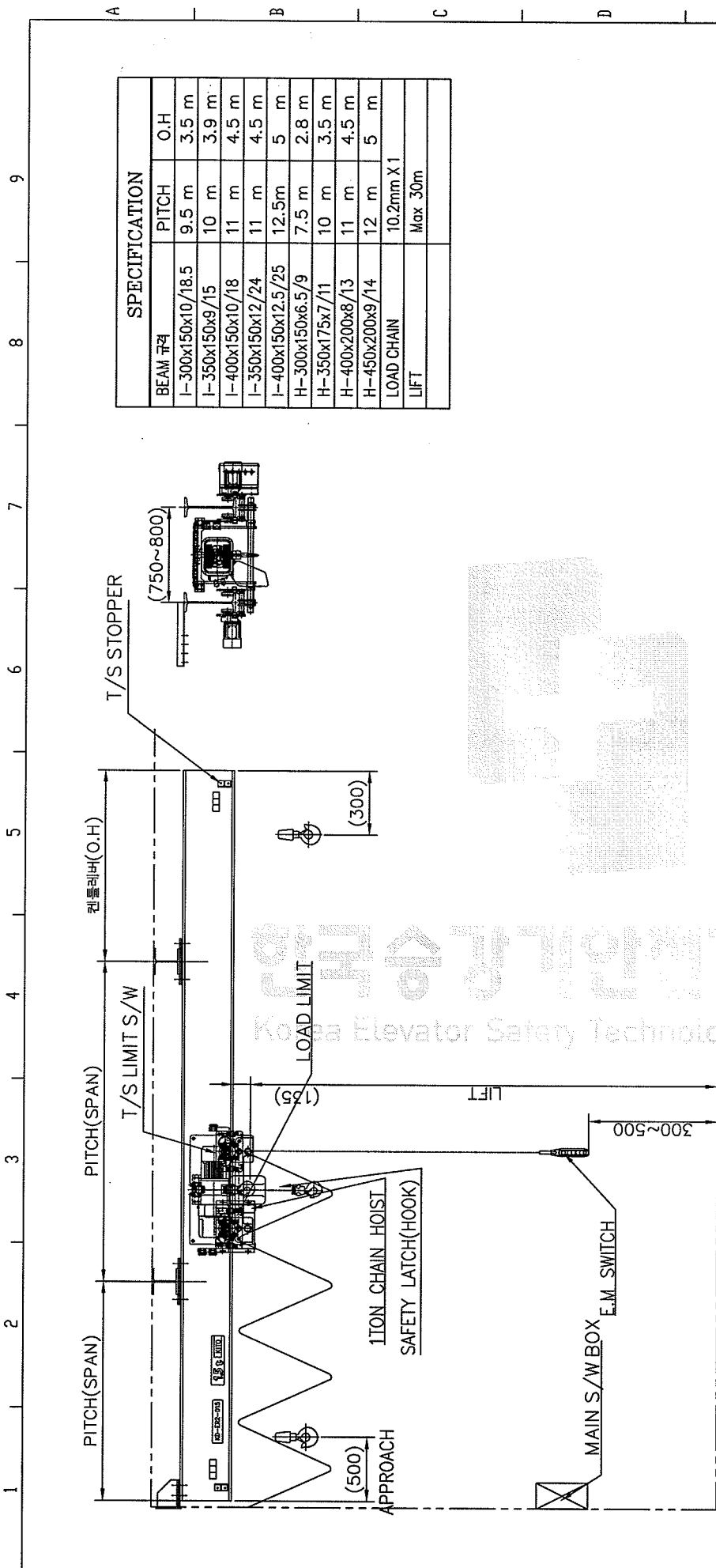
NOTE

1. 재바라 용선
2. DUSTPAN 옵션

Particulars	ER2-E
Model No.	1.5t
Capacity	5t (Max. 30m)
LIFT	φ10.2 x 1
Chain	150 mm
Motor	3φ 220V/200V 60Hz 380, 440V 60Hz
Power	1.8kW x 4P
Motor Output	0.4kW x 4P x 2EA
Speed	S 5.4 m/min IS 5.3/4.9 m/min IL 12/2 m/min IS 24/4 m/min L 12 m/min S 24 m/min
Stroke	5m (Max. 30m)
Length of Power Supply Cable	(2) m
Beam Width (150~200)	255mm ~ MAX (445)mm
Beam Width	150 175 200
Flange Width	154mm 179mm 204mm
Weight	Approx. 355kg
Painting Color	Munsell 7.5R7/14

承認	設計	製圖	名 稱
APPROVED	DESIGNED	DRAWN	TITLE
		W.H.E	1.5t ER2M SERIES ELECTRIC CHAIN HOIST WITH MOTORIZED TOROLLEY
承認	設計	製圖	尺 度
APPROVED	DESIGNED	DRAWN	SCALE
			NOT
改訂	年 月 日	承認	圖 番
REV.	DATE	APPROVAL	DWG NO.
			KK-ER2-015-A-002
			実 買 回 数
			REV. 0

株式会社 KITO CORP.



SPECIFICATION

BEAM 규격	PITCH	O.H
I-300x150x10/18.5	9.5 m	3.5 m
I-350x150x9/15	10 m	3.9 m
I-400x150x10/18	11 m	4.5 m
I-350x150x12/24	11 m	4.5 m
I-400x150x12.5/25	12.5m	5 m
H-300x150x6.5/9	7.5 m	2.8 m
H-350x175x7/11	10 m	3.5 m
H-400x200x8/13	11 m	4.5 m
H-450x200x9/14	12 m	5 m
LOAD CHAIN	10.2mm X 1	
LIFT	Max 30m	

NAME PLATE

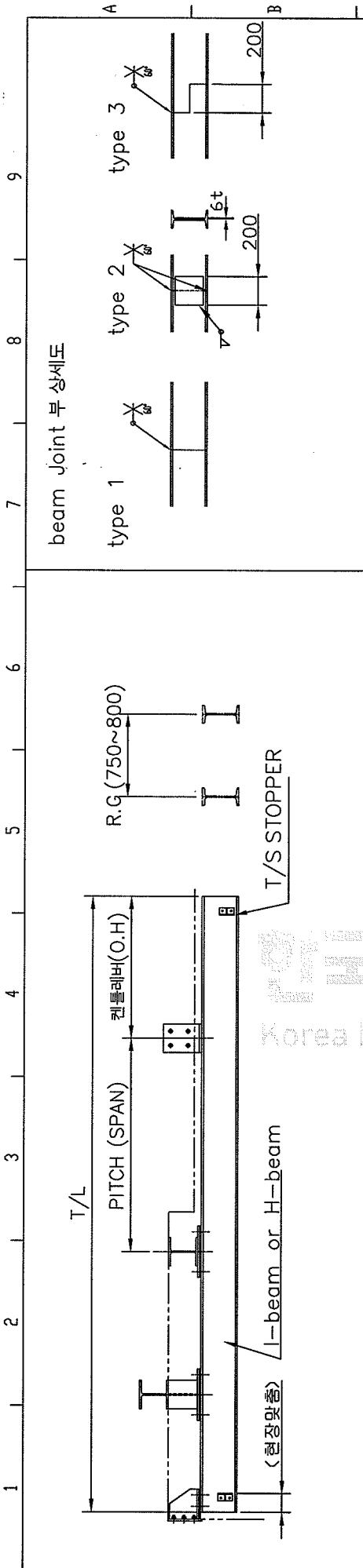
장 려 하 중	1.5 ton
전기 설비 정격	V
제 조 자	
제 조 년 월	20 . .
안전인증 표시	IS
형 식 번 호	KD-ER2-015
제 조 번 호	

형식번호: KD-ER2-015	형식번호: KD-ER2-015
Model number. KITO-C-ER2D015S-S KITO-C-ER2D015S-L KITO-C-ER2D015S-IS KITO-C-ER2D015S-IL	Model number. KITO-ER2D015S-S KITO-ER2D015S-L KITO-ER2D015S-IS KITO-ER2D015S-IL
KITO-C-ER2D015S-S KITO-C-ER2D015S-L KITO-C-ER2D015S-IS KITO-C-ER2D015S-IL	KITO-ER2D015S-S KITO-ER2D015S-L KITO-ER2D015S-IS KITO-ER2D015S-IL

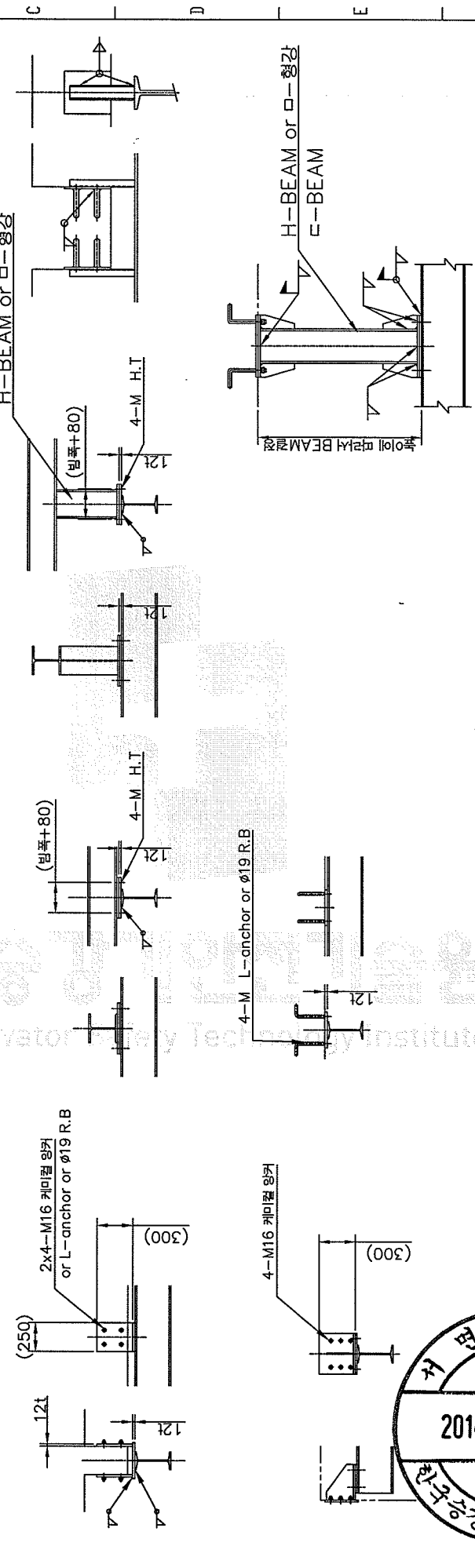
제 목 TITLE		1.5t MONO RAIL HOIST GENERAL ASSEMBLY-1	
圖樣號碼 CODE	圖樣 INVG.NL	尺碼 SCALE	變更回數 REV.
	MONO RAIL 1		

承認 APPROVED	檢査 CHECKED	設計 DESIGNED	圖樣 DRAWN
J.S. CHO	J.S. CHO	EUNWON HEE	EUNWON HEE
株式会社 KATO CORP.			
承認 APPROVED	檢査 CHECKED	設計 DESIGNED	圖樣 DRAWN
年.月.日 DATE	承認 APPROVED	檢査 CHECKED	圖樣 DRAWN
CONTENTS			
REV.	REV.	REV.	REV.



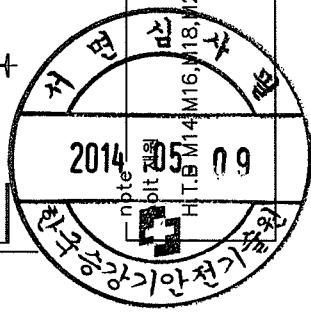


beam 과 상부와의 연결종류



용접재질
 각장은 높이 5일때 용접길이 L=60mm 이상
 각장은 높이 6일때 용접길이 L=50mm 이상
 각장은 높이 7일때 용접길이 L=40mm 이상

용접재질
 각장은 높이 5일때 용접길이 L=60mm 이상
 각장은 높이 6일때 용접길이 L=50mm 이상
 각장은 높이 7일때 용접길이 L=40mm 이상



특기사항
 1. 지시없는 용접부는 용접모재의 70%로 용접한다.

REV.	DATE	APPROVED	CHECKED	DESIGNED	DRAWN	TITLE	SCALE
01		J.S. CHO	J.S. CHO	EDUNWON HEE	EDUNWON HEE	1.5t MONO RAIL HOIST GIRDER BEAM	1:1
02							
03							
04							
05							
06							
07							
08							
09							
10							

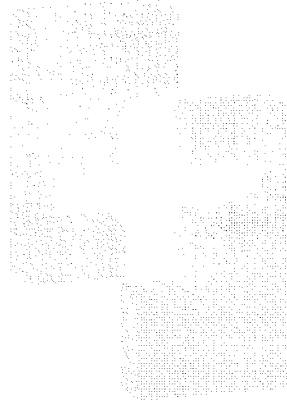
CONTENTS

MONO RAIL HOIST GIRDER BEAM

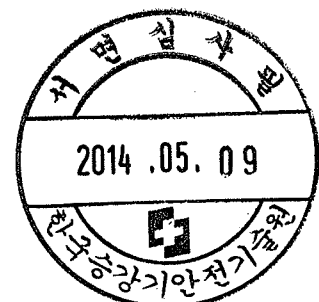
MONO RAIL-D1

4. 전 기 도 면

- 1) ELECTRICAL SPECIFICATION
- 2) SYMBOL LIST
- 3) 배선배관도 & 접지계통도
- 4) 전기회로도
- 5) PANEL 관련도



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Korea Elevator Safety Technology Institute



LOAD SUMMARY 1 (ER2-015IS-IL/IS)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	11.2 (A)	6 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 17.7 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 17.7 * 1.25 = 22.1 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	5.1 (A)	5 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

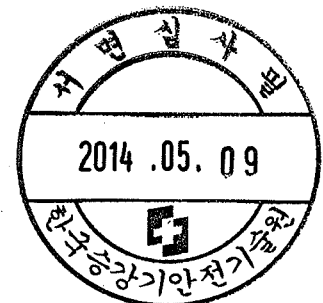
*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 10.6 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 10.6 * 1.25 = 13.2 A



LOAD SUMMARY 2 (ER2-015IS-L/S)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	11.2 (A)	6 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 17.7 A

*** PEAK 전류값 ***

K= NOMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 17.7 * 1.25 = 22.1 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	5.1 (A)	4.4 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

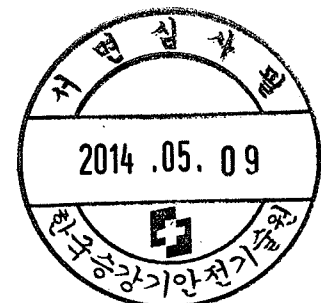
*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 10 A

*** PEAK 전류값 ***

K= NOMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 10 * 1.25 = 12.5 A



LOAD SUMMARY 3 (ER2-015S-IL/IS)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	8.4 (A)	6 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 14.9 A

*** PEAK 전류값 ***

K= NOMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 14.9 * 1.25 = 18.6 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	4.6 (A)	5 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

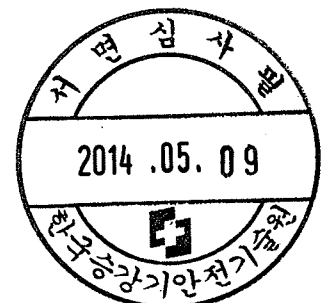
*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 10.1 A

*** PEAK 전류값 ***

K= NOMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 10.1 * 1.25 = 12.6 A



LOAD SUMMARY 4 (ER2-015S-L/S)

*POWER SOURCE : AC 3Φ 220(208)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	8.4 (A)	6 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 14.9 A

*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용

NOMAL 전류값 * K = 14.9 * 1.25 = 18.6 A

*POWER SOURCE : AC 3Φ 380(440)V

OBJECT	HOISTING	TRAVERSING	CONTROL CIRCUIT
MOTOR OUTPUT	1.8KW x 4P	0.4KW x 4P x 2SET	
FULL LOAD CURRENT	4.6 (A)	4.4 (A)	0.5 (A)

*크레인 하중상태를 HOIST의 정격 LOAD의 100(%)를 사용했을때를 기준으로 작성하였음.

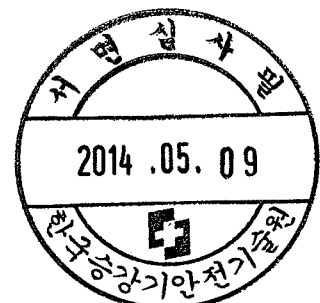
*** NOMAL 전류값 ***

권상과 횡행시 : HOISTING + TRAVERSING + CONTROL CIRCUIT = 9.5 A




*** PEAK 전류값 ***

K= NAMAL 전류치가 50A미만일때 1.25, 50A이상일때 1.1적용





NOMAL 전류값 * K = 9.5 * 1.25 = 11.8 A






ROTATING MACHINE

-  SYNCHRONOUS GENERATOR, 3-PHASE
-  AC INDUCTION MOTOR, 3-PHASE
* N : NORMAL DUTY
S : STAND-BY
-  DC MOTOR





LIGHTNING ARRESTERS

-  LA : LIGHTNING ARRESTER
-  SA : SURGE ARRESTER
-  SS : SURGE SUPPRESSOR
-  DISCHARGE COUNTER

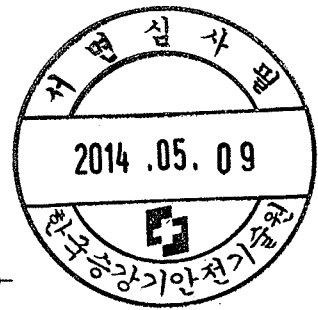
INSTRUMENT TRANSFORMERS

-  CURRENT TRANSFORMER
-  ZERO PHASE CURRENT TRANSFORMER
-  POTENTIAL TRANSFORMER
























CIRCUIT BREAKERS

-  POWER CIRCUIT BREAKER, FIXED TYPE
-  GCB : SF6 GAS CIRCUIT BREAKER
-  VCB : VACUUM CIRCUIT BREAKER
-  ACB : AIR CIRCUIT BREAKER






POWER CIRCUIT BREAKER, DRAWOUT TYPE
























SWITCHES

-  CIRCUIT BREAKER, FIXED TYPE
-  MCCB : MOULDED CASE CIRCUIT BREAKER
-  MCB : MINIATURE CIRCUIT BREAKER
-  CIRCUIT BREAKER, DRAWOUT TYPE
-  WITHDRAWABLE INTERCONNECTOR
-  CIRCUIT BREAKER, MANUALLY OPERATED FIXED TYPE WITH THERMAL & MAGNETIC TRIP
-  CIRCUIT BREAKER, MANUALLY OPERATED FIXED TYPE WITH MAGNETIC TRIP ONLY
-  CIRCUIT BREAKER, MANUALLY OPERATED DRAWOUT TYPE WITH THERMAL & MAGNETIC TRIP
-  CIRCUIT BREAKER, MANUALLY OPERATED FIXED TYPE WITH THERMAL & MAGNETIC TRIP AND RESIDUAL CURRENT RELEASE
-  DISCONNECTOR SWITCH, SINGLE THROW MANUALLY OPERATED
-  LOAD BREAK SWITCH, SINGLE THROW MANUALLY OPERATED,
-  EARTHING SWITCH, SINGLE THROW MANUALLY OPERATED
-  DISCONNECTOR SWITCH, SINGLE THROW MOTOR OPERATED
-  EARTHING SWITCH, SINGLE THROW MOTOR OPERATED
-  VACUUM CIRCUIT SWITCH
-  FUSED DISCONNECTOR SWITCH
-  FUSE-SWITCH
-  LIMIT SWITCH (MAKE CONTACT)
-  LIMIT SWITCH (BREAK CONTACT)
-  PUSH BUTTON, NORMALLY OPEN MOMENTARY CONTACT
-  PUSH BUTTON, NORMALLY CLOSED MOMENTARY CONTACT
-  PUSH BUTTON, NORMALLY OPEN PUSH TO LOCK, RELEASED BY KEY
-  MANUAL SELECTOR SWITCH (LOCKED)

CONTACTORS AND STARTERS

-  AUX. CONTACT, NORMALLY OPEN WHEN MAIN SWITCHING DEVICE IS DE-ENERGIZED
-  AUX. CONTACT, NORMALLY CLOSED WHEN MAIN SWITCHING DEVICE IS DE-ENERGIZED
-  MAGNETIC CONTACTOR, ELECTRICALLY OPERATED
-  COMBINATION STARTER, FULL VOLTAGE, NON-REVERSING, DRAWOUT TYPE, WITH ELECTRICALLY OPERATED CONTACTORS, WITH MAGNETIC MOTOR CIRCUIT BREAKER, BUILT IN ELECTRONIC OVER-CURRENT RELAY WITH ADJUSTABLE TRIP RATING
-  COMBINATION STARTER, FULL VOLTAGE, NON-REVERSING, FIXED TYPE, WITH ELECTRICALLY OPERATED CONTACTORS, WITH MAGNETIC MOTOR CIRCUIT BREAKER, BUILT IN THERMAL OVER-CURRENT RELAY WITH ADJUSTABLE TRIP RATING

GRAPHIC SYMBOLS

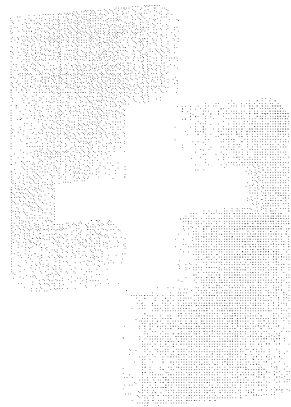
-  GENERAL OPERATING COIL
-  CAPACITOR
-  CAPACITOR VOLTAGE TRANSFORMER(CVT)
-  RESISTOR
-  DIODE
-  CONTROLLED RECTIFIER
-  DC-DC CONVERTER
-  RECTIFIER, BATTERY CHARGER
-  DC-AC INVERTER
-  BATTERY BANK
-  ELECTRIC HEATER, INDICATE 1st OR 3rd AND I.W. RATING, UNLESS OTHERWISE SPECIFIED, TO BE REGARDED AS 1st.
-  EARTHING CONNECTION
-  DISCONNECTION LINK
-  CROSSING OF CONDUCTORS NOT CONNECTED
-  JUNCTION OF CONDUCTORS OR WIRES
-  BUS DUCT
-  SPB : SEGREGATED PHASE BUS DUCT
-  IPB : ISOLATED PHASE BUS DUCT
-  CABLE HEAD AND CABLE CONNECTION
-  AMMETER SWITCH
-  VOLTMETER SWITCH

- SIGNAL LAMP
- * R = RED
- G = GREEN
- W = WHITE
- A = AMBER
- C = CYAN

SYMBOL LIST

APPROVED	CHECKED	DESIGNED
KOTO CORP.		
	CODE	SCALE
	DWG. NO.	DATE
	SYMBOL LIST	

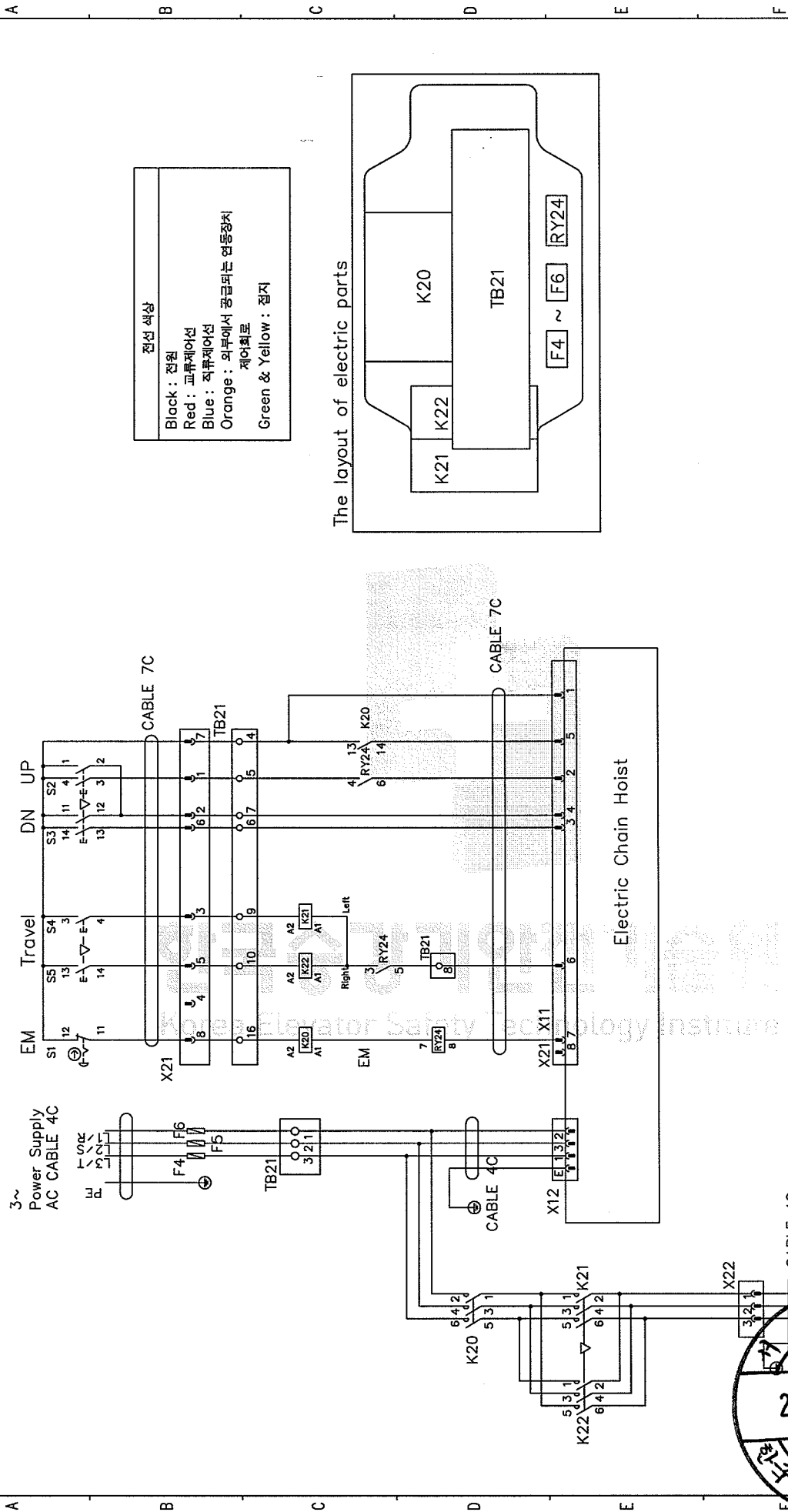
. 1속형 hoisting/. 1속형 traversing



한국승강기안전기술원
Korea Elevator Safety Technology Institute

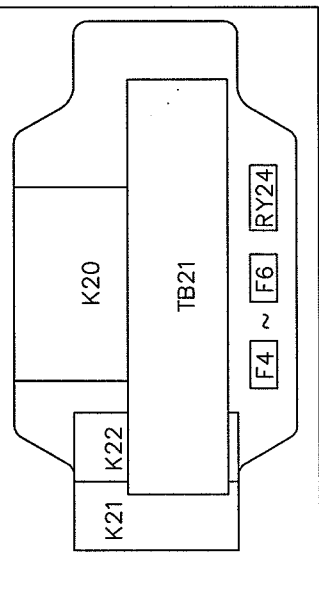


10 9 8 7 6 5 4 3 2 1



전선 색상
 Black : 전선
 Red : 교류제어선
 Blue : 직류제어선
 Orange : 외부에서 공급되는 영동장치 제어회로
 Green & Yellow : 접지

The layout of electric parts



특수 공구(+, 육각)을 사용하여
 개방구 쪽은 제외

APPROVED	CHECKED	DESIGNED	Traversing 1speed Wiring Diagram	
H.Furiya	H.Hatano	K.Miwa	CODE	SCALE
09 / 04 / 21	09 / 04 / 21	09 / 04 / 21	DWG.NO. SEWG3DD0101	DATE

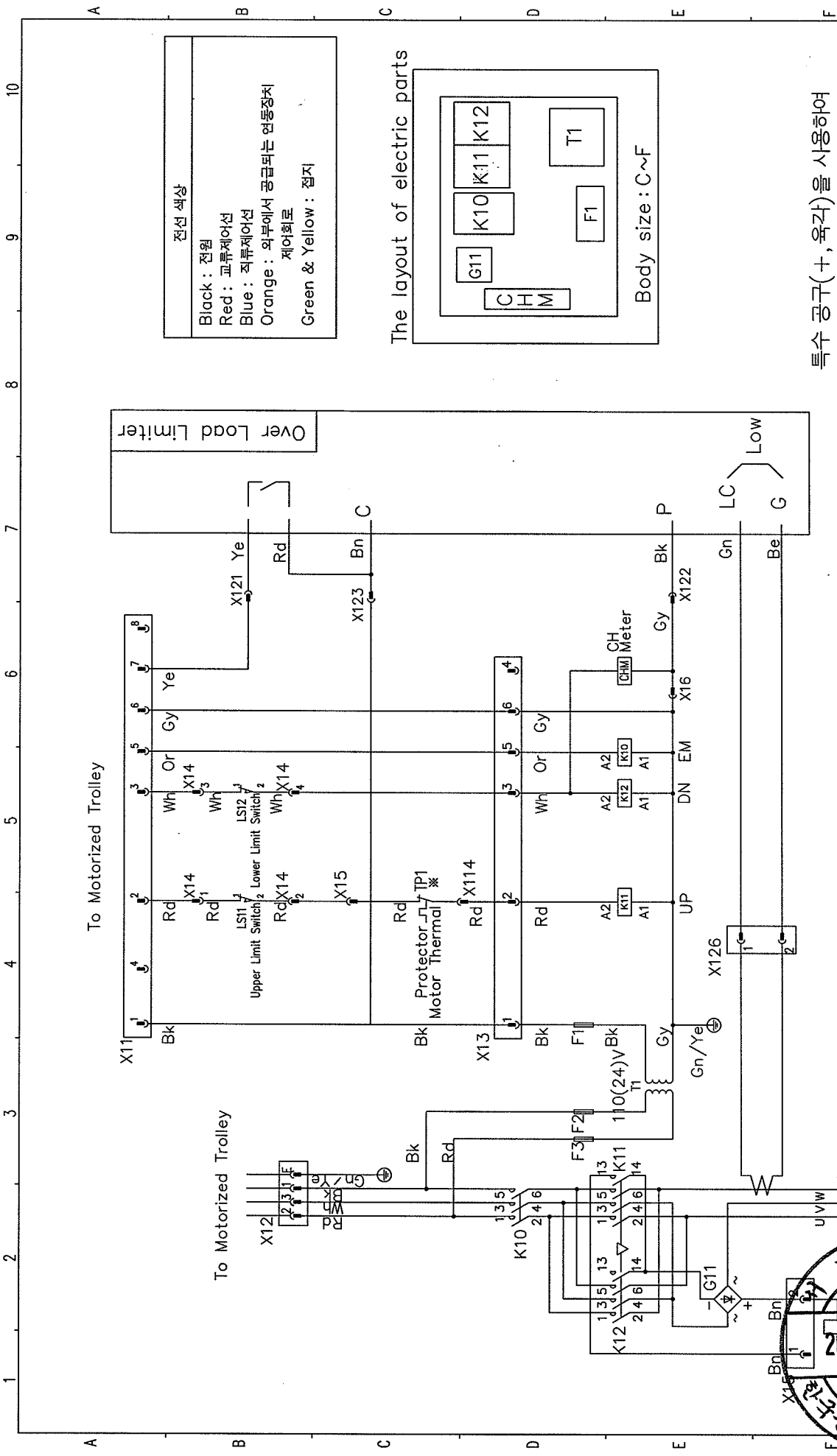
UNIT : mm

2014 . 05 . 09

한국산업기술대학교

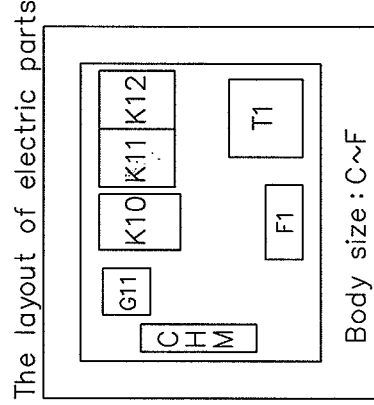
Motor Assy

O.4KwX2SET
 M21
 Traversing



전선 색상

Black : 전원
Red : 교류제어선
Blue : 직류제어선
Orange : 외부에서 공급되는 운동장치 제어회로
Green & Yellow : 접지



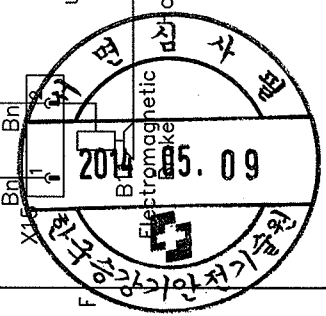
특수 공구(+, 육각)을 사용하여
 개방구 쪽은 제외
 For MR connection

APPROVED	CHECKED	DESIGNED
H.Furiya 09/04/21	T.Hatano 09/04/21	K.Miwa 09/04/21

HOIST CORP

Hoisting 1speed Traversing Wiring Diagram	CODE	SCALE	DATE
	SEWCG3100L01_MR	—	—

* TP1 is attached only to 380V and 440V.

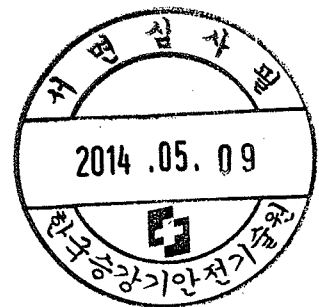


UNIT : mm

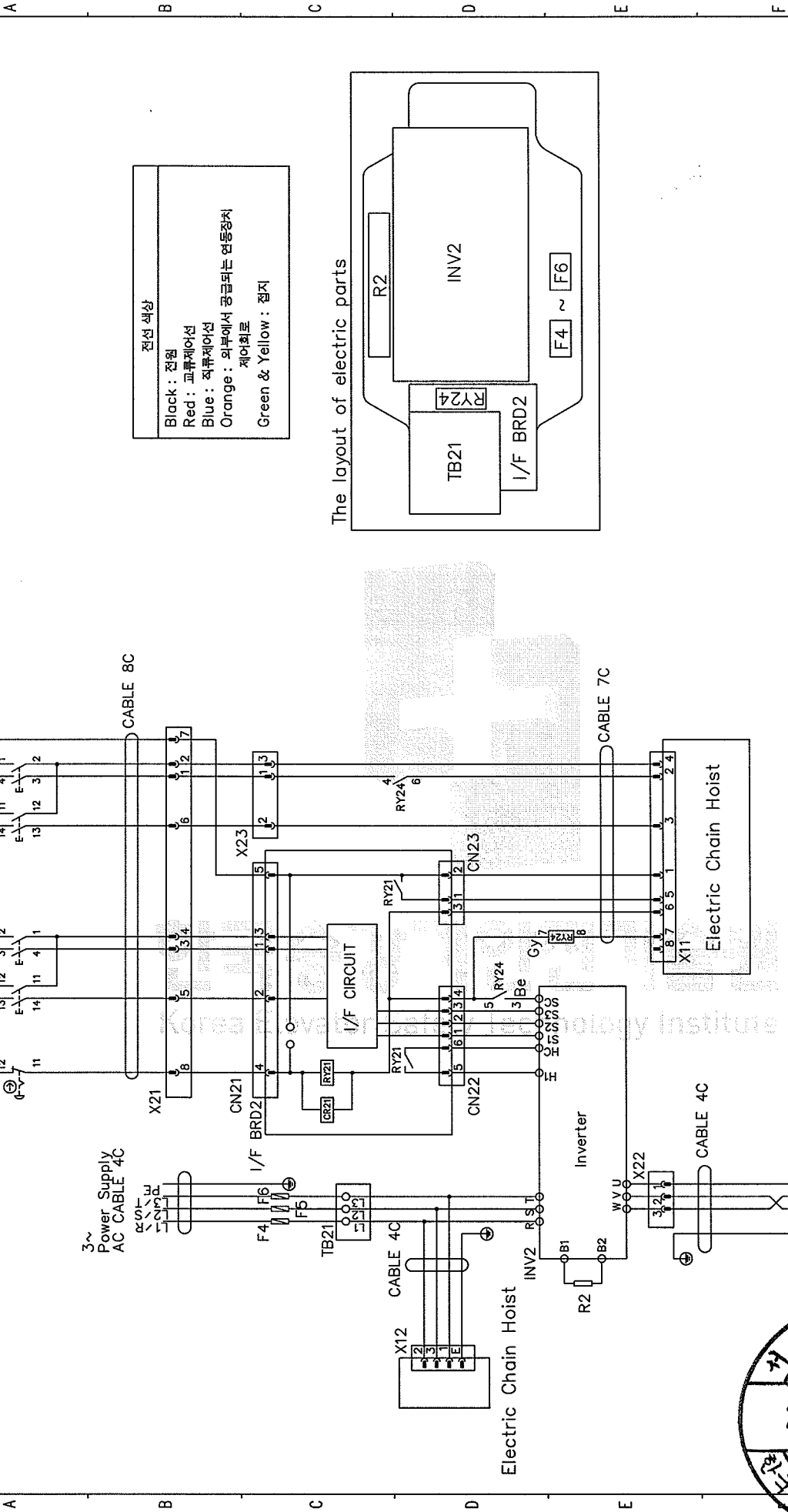
. 1속형 hoisting/. 2속형 traversing



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Korea Elevator Safety Technology Institute

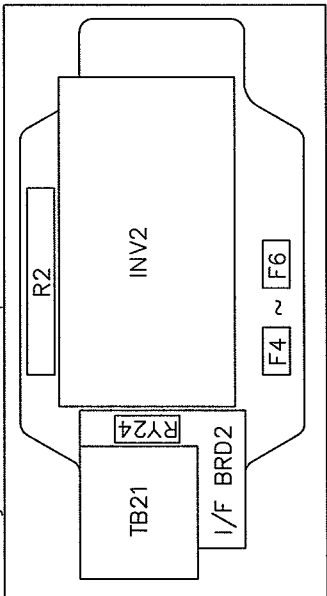


10 9 8 7 6 5 4 3 2 1



전선 색상
 Black : 진월
 Red : 교류제어선
 Blue : 직류제어선
 Orange : 외부에서 공급되는 연동장치 제어회로
 Green & Yellow : 접지

The layout of electric parts



특수 공구(+, 육각)을 사용하여
 개방구 쪽은 제외

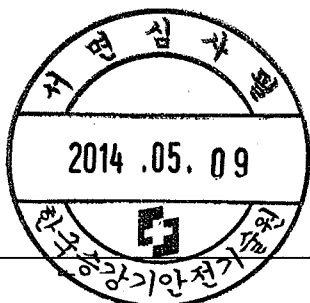
Traversing 2speed
 Wiring Diagram

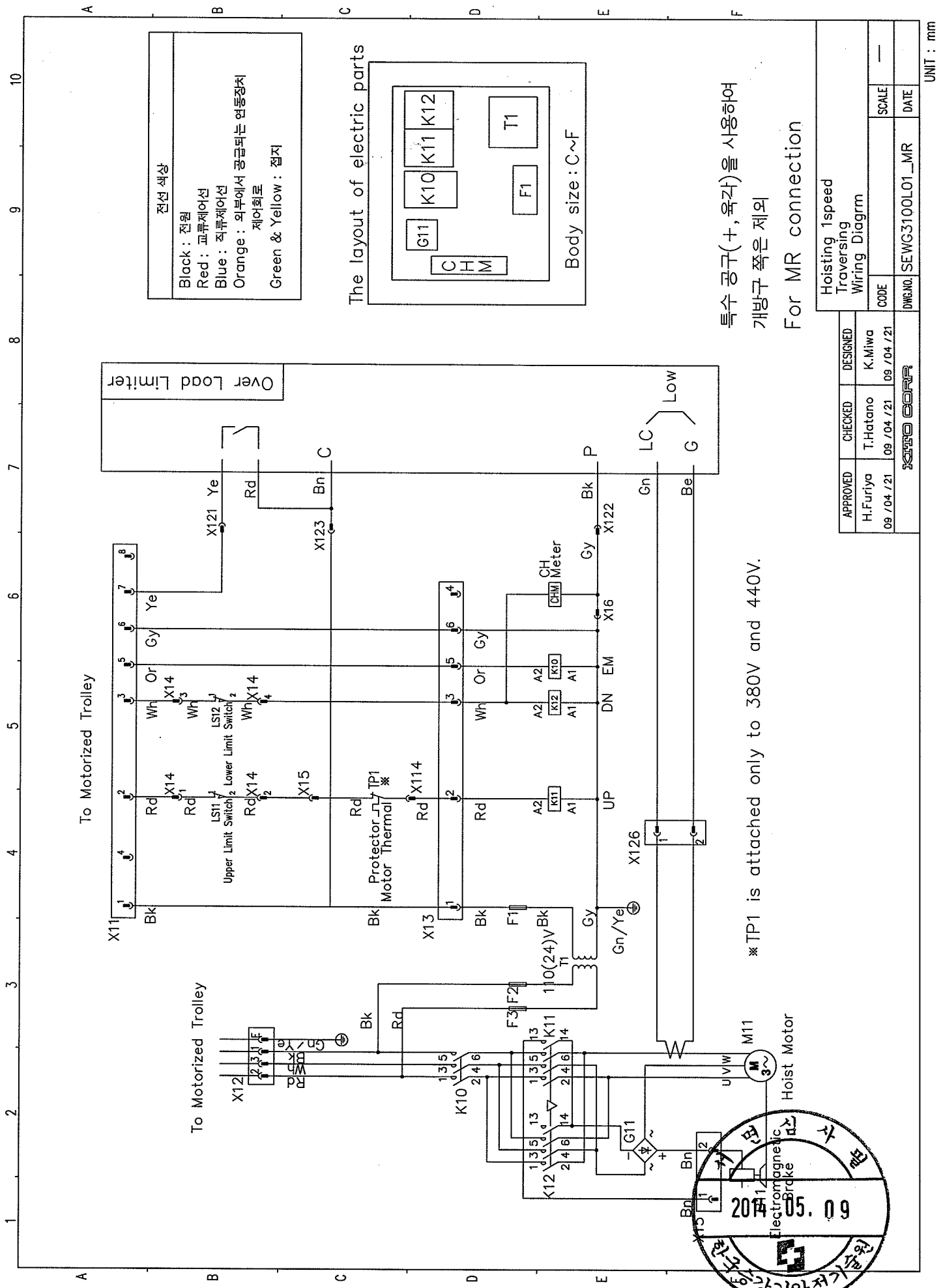
APPROVED	CHECKED	DESIGNED
H.Furiya 09 / 04 / 21	H.Hatano 09 / 04 / 21	K.Miwa 09 / 04 / 21

CODE	SCALE	DATE
---	---	---

DWG.NO SEWG3DD0L01

UNIT : mm





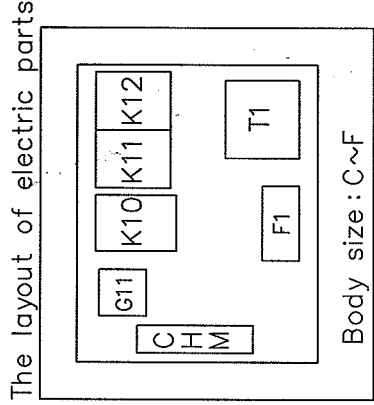
To Motorized Trolley

To Motorized Trolley

*TP1 is attached only to 380V and 440V.

전선 색상

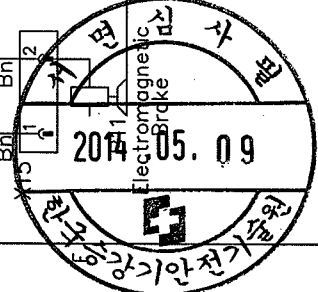
Black : 잔원
Red : 교류제어선
Blue : 직류제어선
Orange : 외부에서 공급되는 연동장치 제어회로
Green & Yellow : 접지



특수 공구(+, 육각)을 사용하여
개방구 쪽은 제외
For MR connection

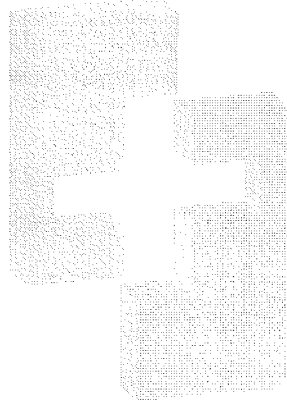
APPROVED	CHECKED	DESIGNED	Hoisting 1speed Traversing Wiring Diagram
H.Furiya 09 /04 /21	T.Hatano 09 /04 /21	K.Miwa 09 /04 /21	CODE SCALE —
			DATE
DWG.NO. SEWC3100L01_MR			

UNIT : mm



5

. 2속형 hoisting/. 1속형 traversing



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Korea Elevator Safety Technology Institute



10

9

8

7

6

5

4

3

2

1

A

B

C

D

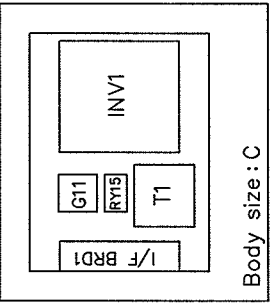
E

F

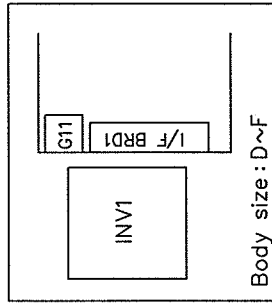
전선 색상

Black : 전원
 Red : 교류제어선
 Blue : 직류제어선
 Orange : 외부에서 공급되는 역동장치 제어회로
 Green & Yellow : 접지

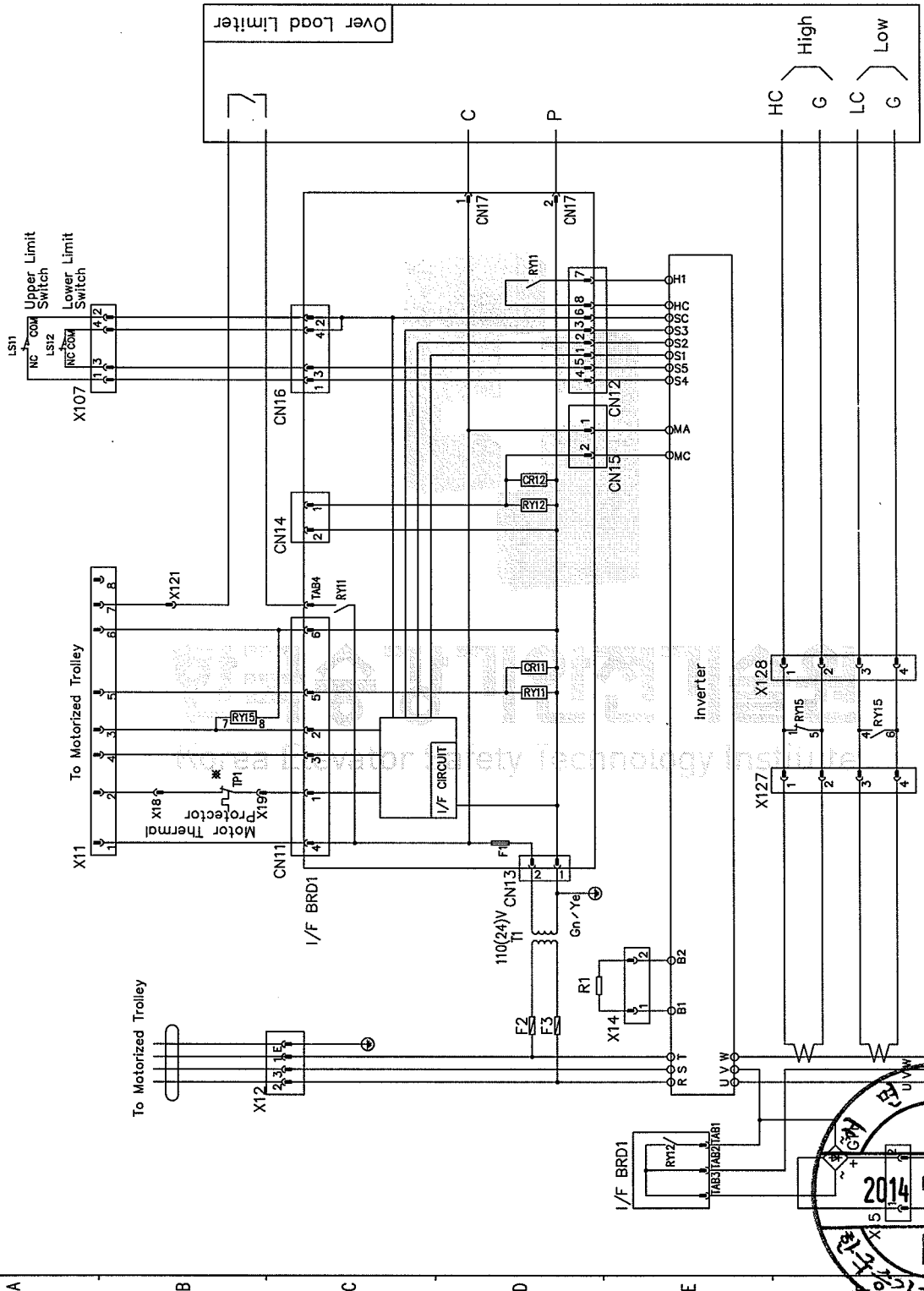
The layout of electric parts



Body size : C



Body size : D~F

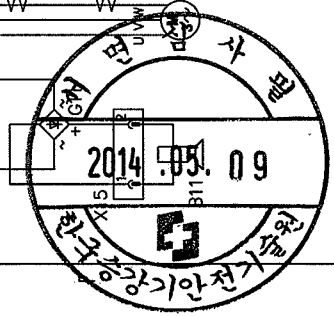


Hoisting 2speed
Traversing
Wiring Diagram

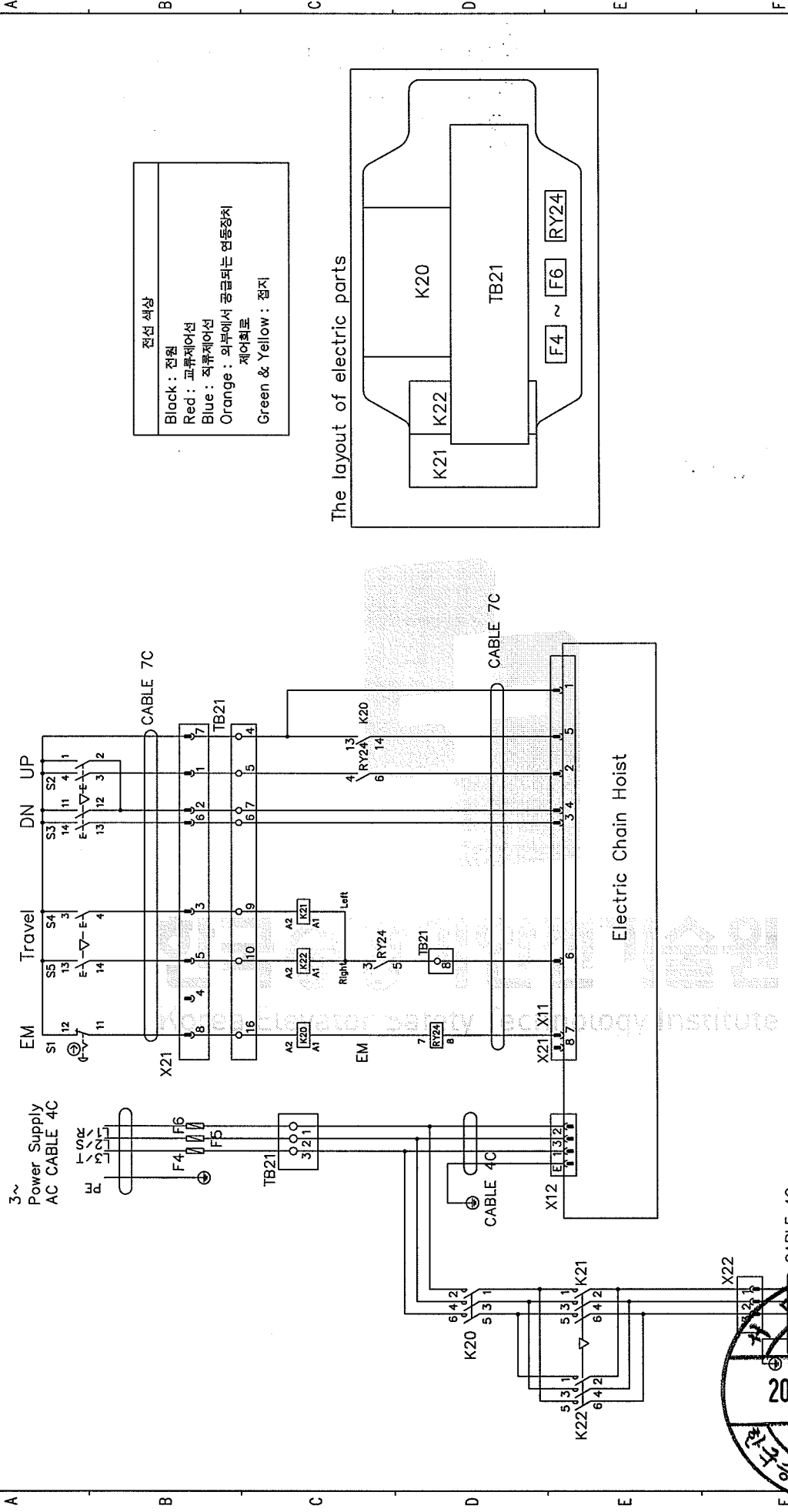
APPROVED	CHECKED	DESIGNED
H.Furiya 09 / 04 / 21	H.Hatano 09 / 04 / 21	K.Miwa 09 / 04 / 21
CODE	SCALE	DATE
DMG.NO. SEWG3D00L01_MR	---	---

UNIT : mm

*TP1 is attached only to 380V and 440V.



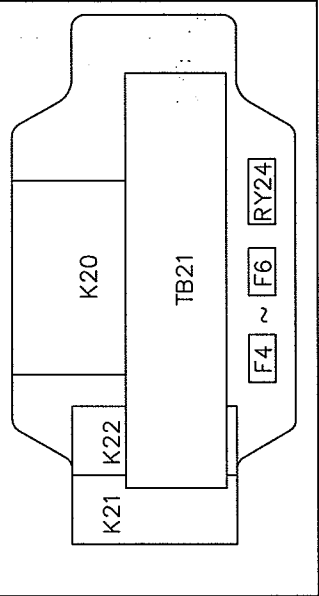
10 9 8 7 6 5 4 3 2 1



전선 색상

Black	: 전선
Red	: 교류제어선
Blue	: 직류제어선
Orange	: 외부에서 공급되는 연동장치 제어회로
Green & Yellow	: 접지

The layout of electric parts

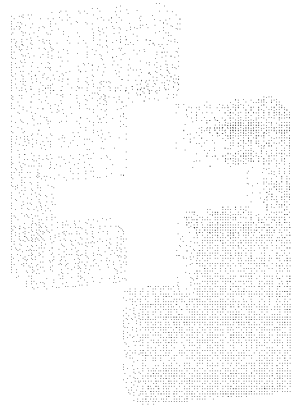


특수 공구(+, 육각)를 사용하여
개방구 쪽은 제외

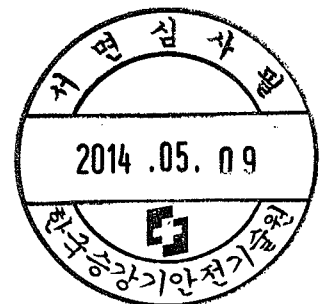
APPROVED		CHECKED		DESIGNED	
H.Furiya		H.Hatano		K.Miwa	
09 / 04 / 21		09 / 04 / 21		09 / 04 / 21	
KATO CORP					
Traversing 1speed Wiring Diagram				CODE	SCALE
DWG.NO. SEWG3DD0L01				DATE	—

UNIT : mm

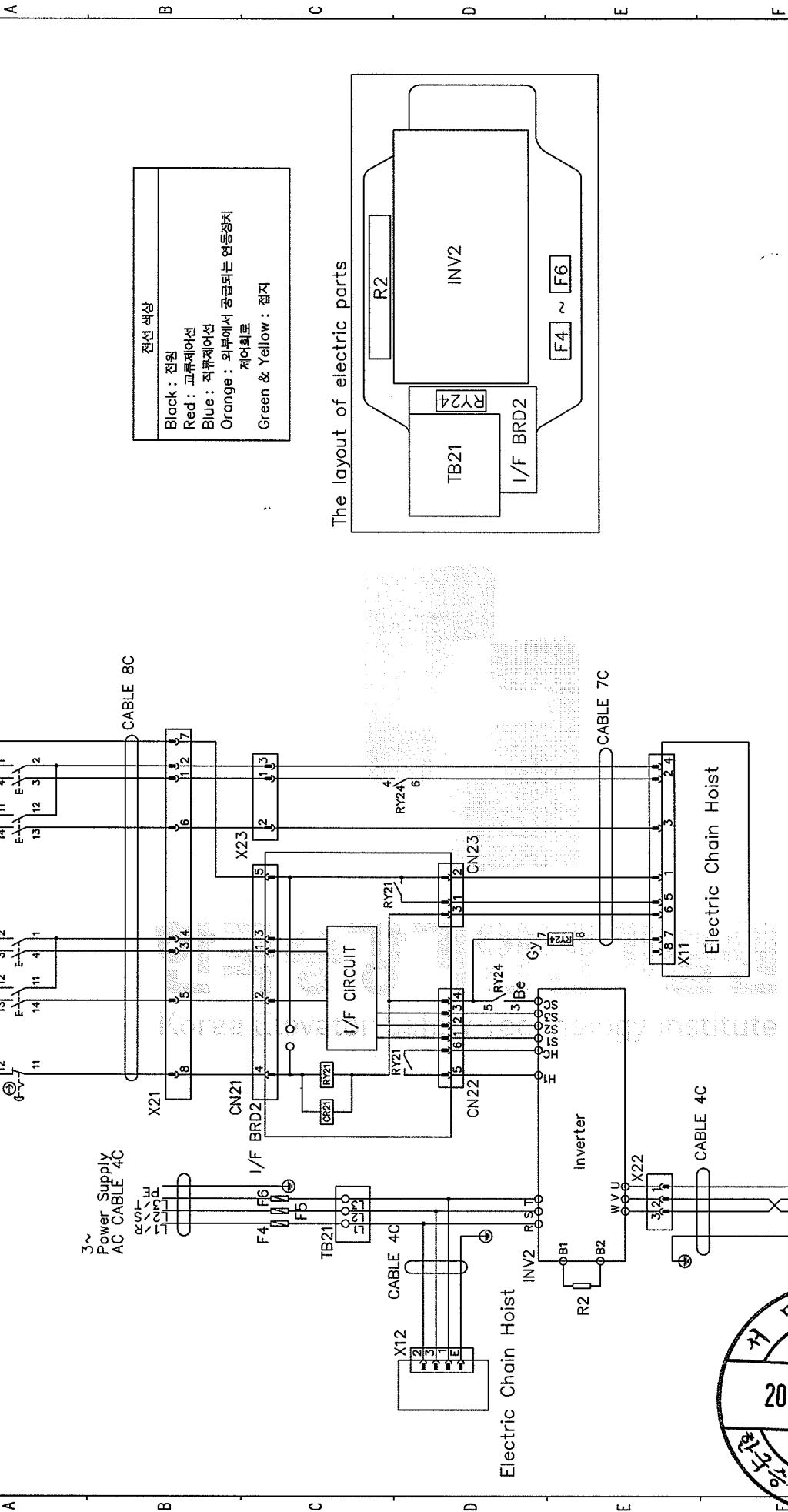
. 2속형 hoisting/. 2속형 traversing



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Korea Elevator Safety Technology Institute

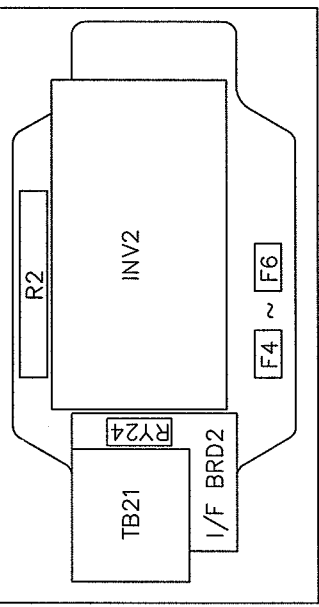


10 9 8 7 6 5 4 3 2 1



전선 색상
 Black : 전선
 Red : 교류제어선
 Blue : 직류제어선
 Orange : 외부에서 공급되는 연동장치 제어회로
 Green & Yellow : 접지

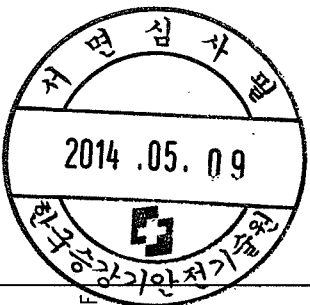
The layout of electric parts



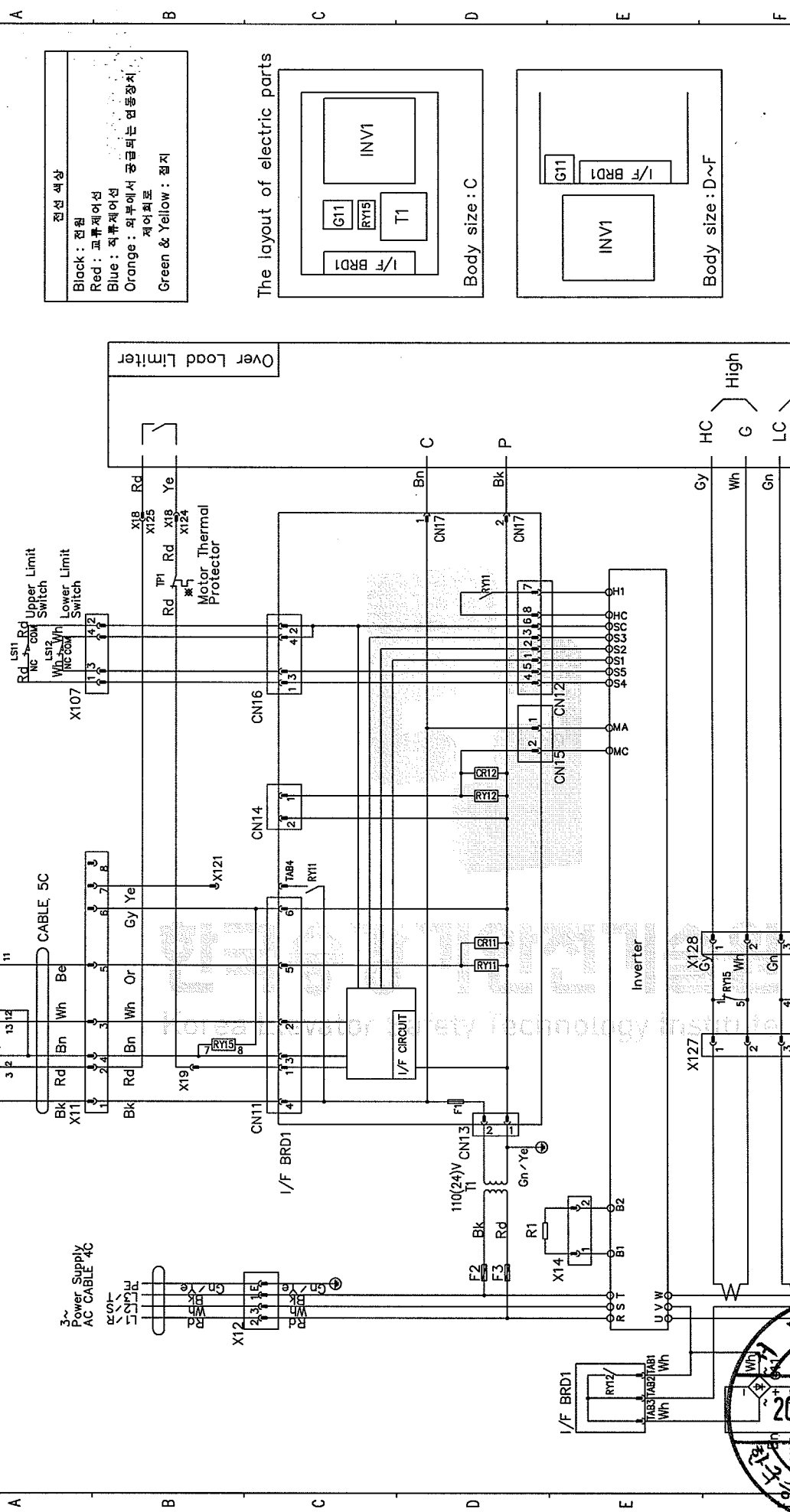
특수 공구(+, 육각)을 사용하여
 개방구 쪽은 제외

APPROVED		CHECKED		DESIGNED	
H.Furiyo		H.Hatano		K.Miwa	
09 / 04 / 21		09 / 04 / 21		09 / 04 / 21	
KATO CORP.					
Traversing 2speed Wiring Diagram				CODE	SCALE
DWG.NO. SEWG3DD0101				DATE	—

UNIT : mm

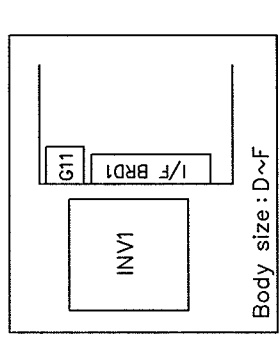
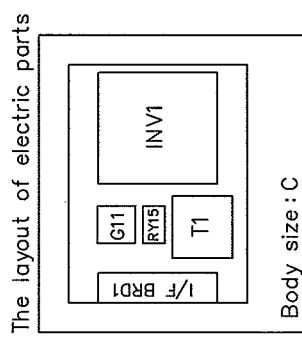


10 9 8 7 6 5 4 3 2 1



전선 색상

Black : 전원
Red : 교류제어선
Blue : 직류제어선
Orange : 외부에서 공급되는 연동장치 제어회로
Green & Yellow : 접지



특수 공구(+, 육각)을 사용하여
개방구 쪽은 제외

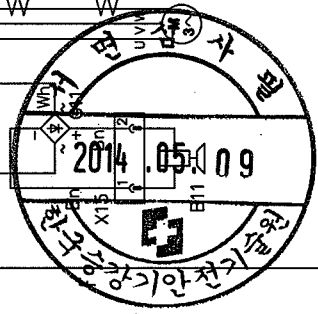
Hoisting 2speed
Wiring Diagram

APPROVED	CHECKED	DESIGNED
H.Furiya	H.Hatano	K.Miwa
09 / 04 / 21	09 / 04 / 21	09 / 04 / 21
CODE	SCALE	DATE
DWG.NO. SEWC3D00L01		

UNIT : mm

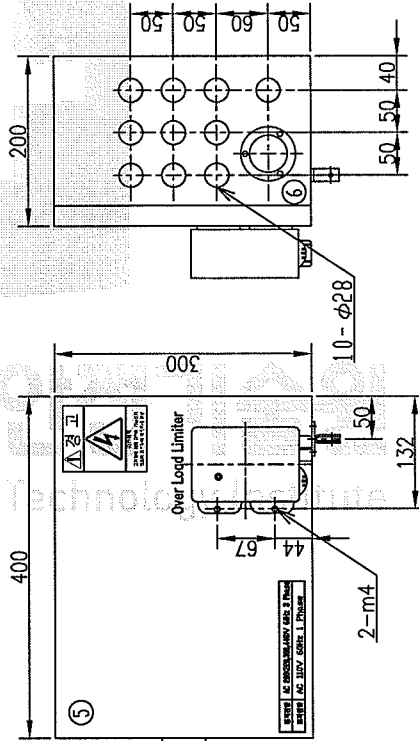
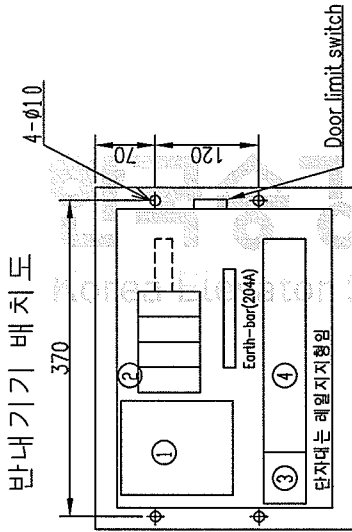
ISTO CORP		
APPROVED	CHECKED	DESIGNED
H.Furiya	H.Hatano	K.Miwa
09 / 04 / 21	09 / 04 / 21	09 / 04 / 21
CODE	SCALE	DATE
DWG.NO. SEWC3D00L01		

*TP1 is attached only to 380V and 440V.



압착단자는 전부 절연피복 부착타입을 사용할 것

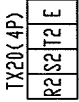
塗装色 : 단셀번호 5Y7/1 (베이커 표준색)
 設定機器 : 인버터



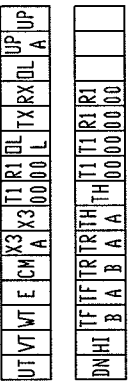
10	削刃加工耐熱時空差	納期
JIS B0405 中級	製作數量	
0.5以上 6以下 ±0.1		
6~25 30, ±0.2		
30, 120, ±0.3		
120, 400, ±0.5		
400, 1000, ±0.8		
1000, 2000, ±1.2		
2000, 4000, ±2.0		

機器番號	名稱	形式	メーカー	個數	備考
1 INV2	인버터	FRNL5C1S-2J21	富士	1	
2	릴레이	HH54P-L (AC24V)	富士	4	
3	소켓	TP514X1	富士	4	
4	단자대	TX20 (4P)	春日	1式	커버부착형
5	합	TX10S (30P)	春日	1式	커버부착형
6	Door limits switch	KH-9015-HL	白象	1	
7			KONDO		
8					
9					
10					

단자대 배열



TX10S (30P)

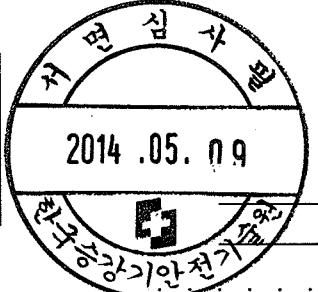


Note

1) 외함 개방 시 충전 부분이 차단되도록 한다.

材質	-	303910
圖番	-	セツゾクハコ
尺 度	尺 NUT	303910-35011
製 造 廠	細田	
設 計	10.10.8	
檢 査	鈴木	
承 認	10.10.8	

樣式 025C-06 三角法 單位: mm



改訂數	內	年 月 日 設 計 承 認
-----	---	---------------

CABLE 구성도 및 사양 - 권상 용량 1.8kW

CABLE SPECIFICATION FOR ER2

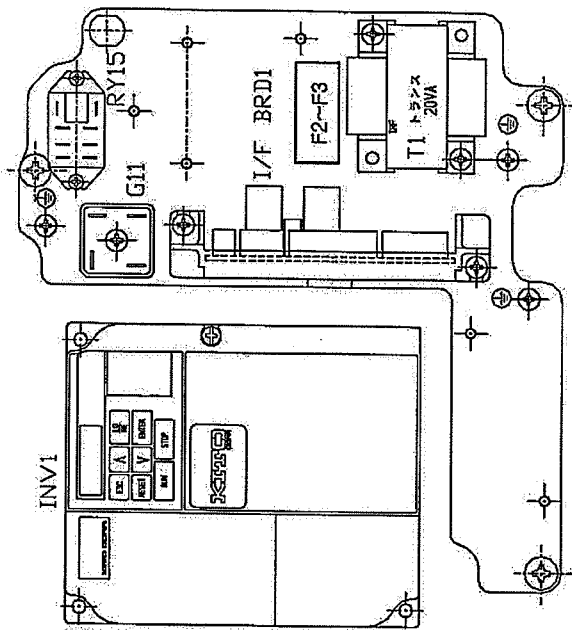
NO	ITEM	TYPE	ER2 015	
			SIZE	
①	Main Power Line(owner scope)	VCT	4sq x 4C	
②	Push Button Switch	VCT	1.25sq x 8C	
③	Loas Limit	VCT	0.75sq x 8C	
④	Power Line for ER	VCT	2sq x 4C	
⑤	Control Line for ER	VCT	1.25sq x 6C	
⑥	Traversing Motor With Earth	VCT	1.25sq x 4C	

(3Φ 220(208)V / 380V / 440V 60HZ)

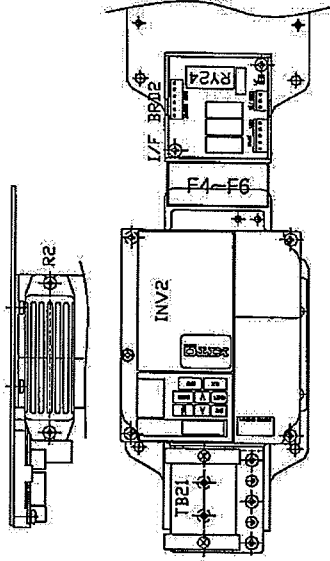


호이스트 CONTROL BOX 배치도 (ER2 015 IS-IL/IS)

HOISTING CONTROL BOX

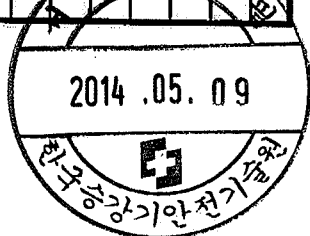


TRAVERSING CONTROL BOX



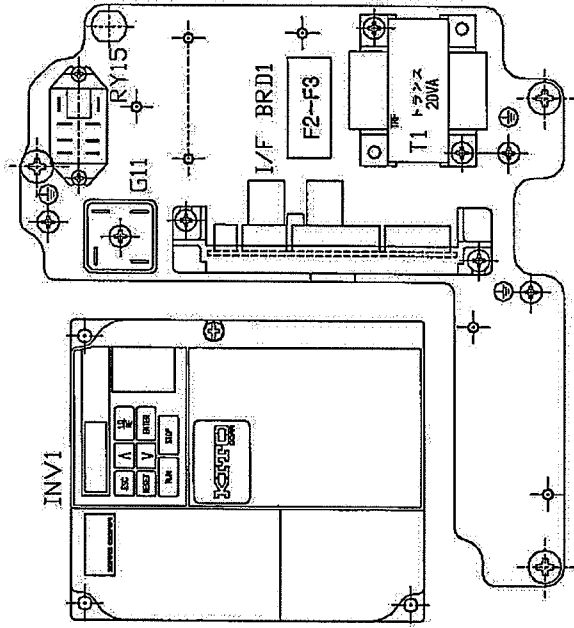
ENCLOSURE : HOIST BODY - IP55
PUSH BUTTON - IP65

MARK	DESCRIPTION	TYPE OF MODEL			Q'TY	MAKER	REMARKS
		220V	380V	440V			
INV1	INVERTER	V1000	V1000	V1000	1	YASKAWA	UP/DOWN
T1	TRANSFORMER	220V/24V 20VA	380V/24V 20VA	440V/24V 20VA	1	KITO	CONTROL CIRCUIT
G11	BRIDGE DIODE	S15VB60	S15VB60	S15VB60	1	SHINDENGEN	
I/F BRD1	INTERFACE BOARD	10~15A	10~15A	10~15A	1	KITO	
F2-F3	GLASS FUSE	10A	10A	10A	2	FUJI	
F4-F6	GLASS FUSE	20A	20A	20A	3	FUJI	
RY15	RELAY	24V	24V	24V	1	OMRON	HIGH/LOW
INV2	INVERTER	V1000	V1000	V1000	1	YASKAWA	RIGHT/LEFT
I/F BRD2	INTERFACE BOARD	10~15A	10~15A	10~15A	1	KITO	
RY24	RELAY	24V	24V	24V	1	OMRON	EMERGENCY STOP
TB21	TERMINAL BOARD 21	10~15A	10~15A	10~15A	1	KITO	

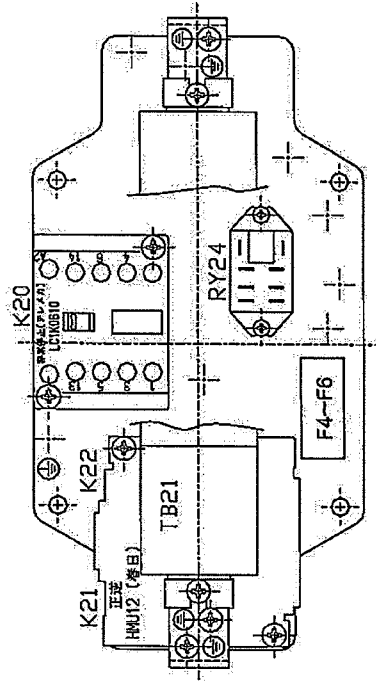


호이스트 CONTROL BOX 배치도 (ER2 015 IS-L/S)

HOISTING CONTROL BOX

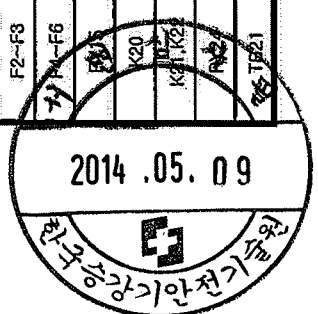


TRAVERSING CONTROL BOX



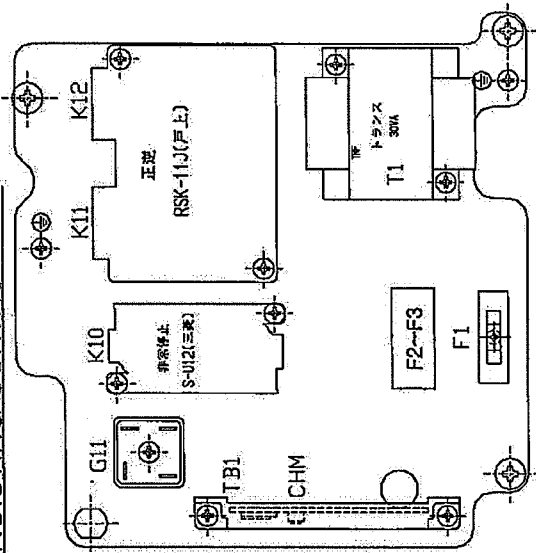
ENCLOSURE : HOIST BODY - IP55
PUSH BUTTON - IP65

MARK	DESCRIPTION	TYPE OF MODEL			Q'TY	MAKER	REMARKS
		220V	380V	440V			
INV1	INVERTER	V1000	V1000	V1000	1	YASKAWA	UP/DOWN CONTROL CIRCUIT
T1	TRANSFORMER	220V/24V 20VA	380V/24V 20VA	440V/24V 20VA	1	KITO	
G11	BRIDGE DIODE	S15VB60	S15VB60	S15VB60	1	SHINDENGEN	
I/F BRD1	INTERFACE BOARD	10~15A	10~15A	10~15A	1	KITO	
F2-F3	GLASS FUSE	10A	10A	10A	2	FUJI	
F4-F6	GLASS FUSE	20A	20A	20A	3	FUJI	
RY15	RELAY	24V	24V	24V	1	OMRON	HIGH/LOW
K20	MAGNET CONTACTOR	LG1K0610B7	LG1K0610B7	LG1K0610B7	1	TELEMECANIQUE	EMERGENCY STOP
K21, K22	MAGNET CONTACTOR	HMU12	HMU12	HMU12	1	KASUGA	RIGHT/LEFT
F4, F5	RELAY	24V	24V	24V	1	OMRON	EMERGENCY STOP
TB21	TERMINAL BOARD 21	10~15A	10~15A	10~15A	1	KITO	

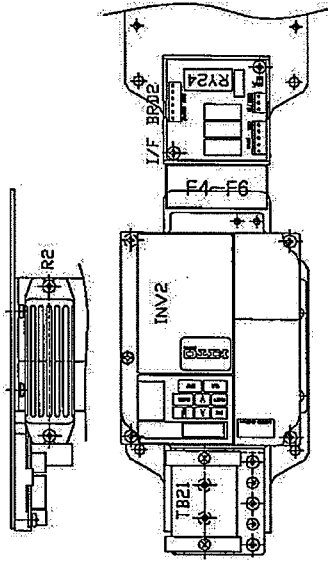


호이스트 CONTROL BOX 배치도 (ER2 015 S-IL/IS)

HOISTING CONTROL BOX

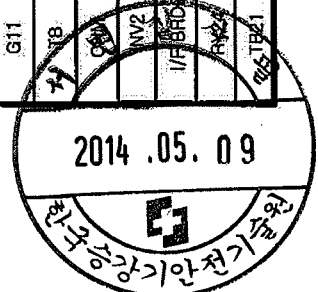


TRAVERSING CONTROL BOX



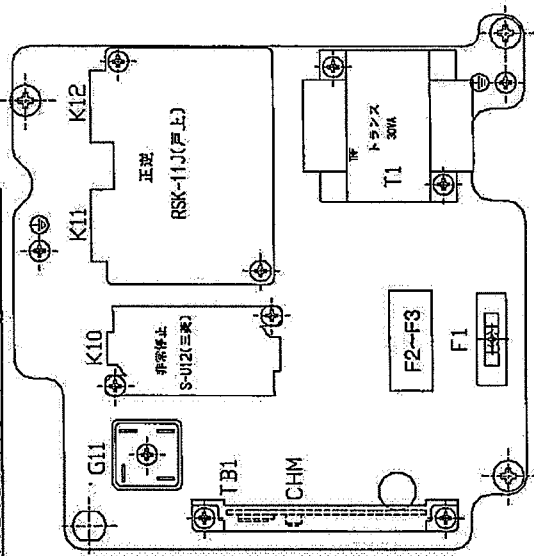
ENCLOSURE : HOIST BODY - IP55
PUSH BUTTON - IP65

MARK	DESCRIPTION	TYPE OF MODEL		Q'TY	MAKER	REMARKS
		220V	380V			
K10	MAGNET CONTACTOR	S-U12	S-U12	1	mitsubishi	EMERGENCY STOP
K11, K12	MAGNET CONTACTOR	RSK-11J-S95	RSK-11J-S95	1	TOGAMI	UP/DOWN
T1	TRANSFORMER	220V/24V 30VA	380V/24V 30VA	1	KITO	CONTROL CIRCUIT
F1	GLASS FUSE	2A	2A	1	FUJII	
F2-F3	GLASS FUSE	10A	10A	2	FUJII	
F4-F6	GLASS FUSE	20A	20A	3	FUJII	
G11	BRIDGE DIODE	S15VB60	S15VB60	1	SHINDENGEN	
TB	TERMINAL BOARD	10-15A	10-15A	1	KITO	
INV2	COUNTER HOUR METER	ECP91CHAA1-3	ECP91CHAA1-3	1	OTEC	사용회수, 시간 기록
I/F BRD	INVERTER	V1000	V1000	1	YASKAWA	RIGHT/LEFT
R2	INTERFACE BOARD	10-15A	10-15A	1	KITO	
R21	RELAY	24V	24V	1	OMRON	EMERGENCY STOP
TB21	TERMINAL BOARD 21	10-15A	10-15A	1	KITO	

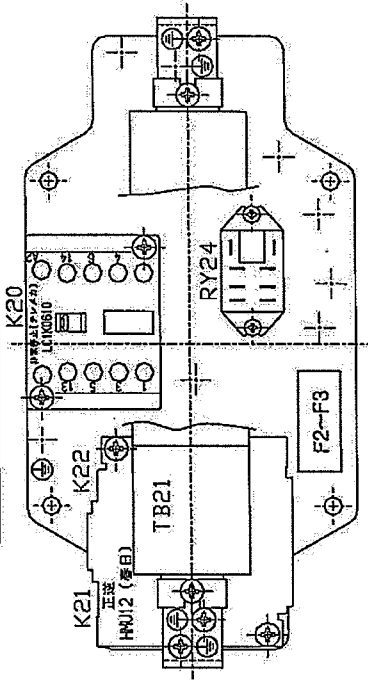


호이스트 CONTROL BOX 배치도 (ER2 015 S-L/S)

HOISTING CONTROL BOX

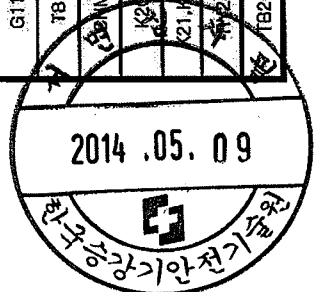


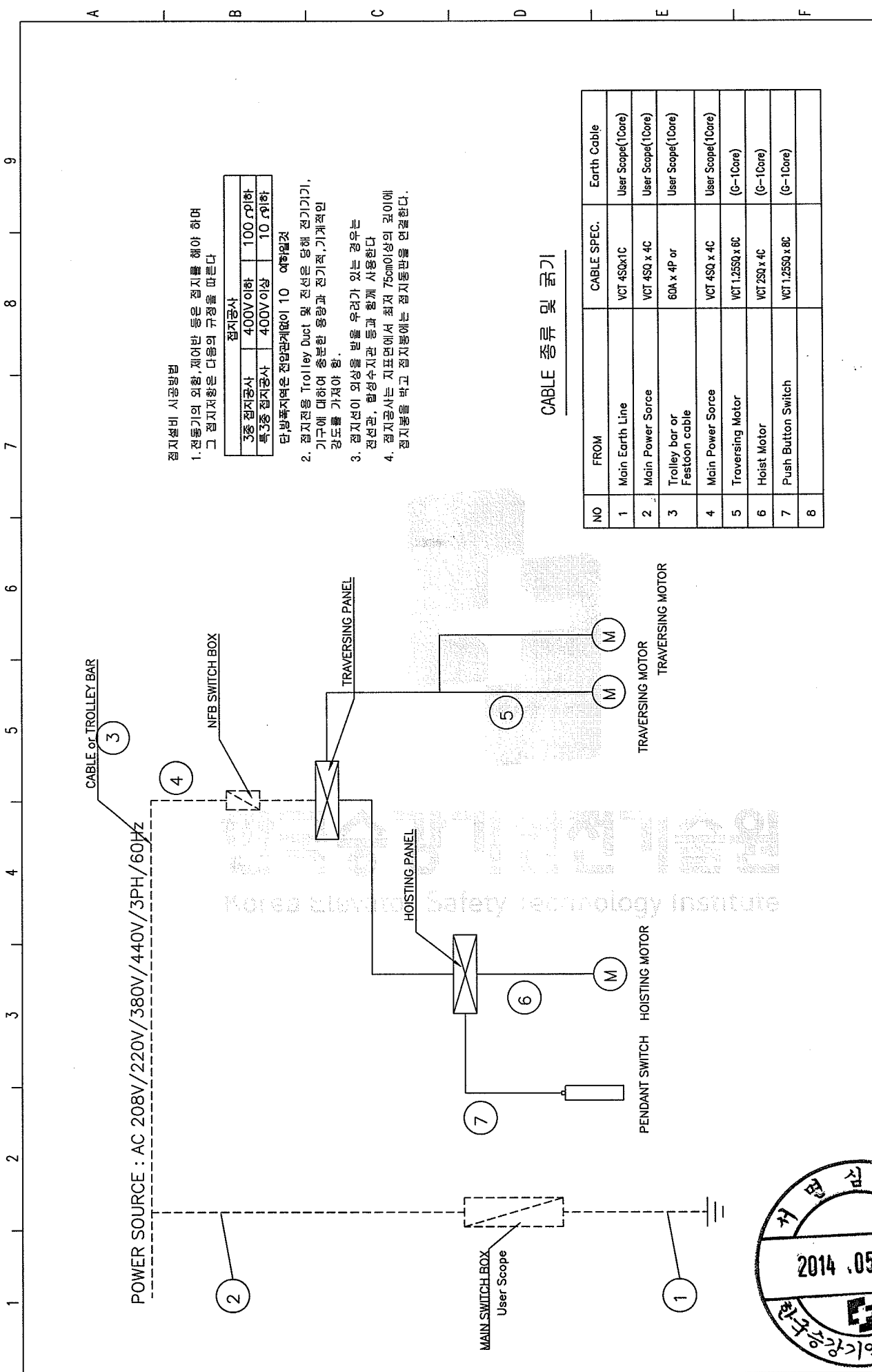
TRAVERSING CONTROL BOX



ENCLOSURE : HOIST BODY - IP55
PUSH BUTTON - IP65

MARK	DESCRIPTION	TYPE OF MODEL			Q'TY	MAKER	REMARKS
		220V	380V	440V			
K10	MAGNET CONTACTOR	S-U12	S-U12	S-U12	1	mitsubishi	EMERGENCY STOP
K11,K12	MAGNET CONTACTOR	RSK-11J-S95	RSK-11J-S95	RSK-11J-S95	1	TOGAMI	UP/DOWN
T1	TRANSFORMER	220V/24V 30VA	380V/24V 30VA	440V/24V 30VA	1	KITO	CONTROL CIRCUIT
F1	GLASS FUSE	2A	2A	2A	1	FUJI	
F2-F3	GLASS FUSE	10A	10A	10A	2	FUJI	
F4-F6	GLASS FUSE	20A	20A	20A	3	FUJI	
G11	BRIDGE DIODE	S15VB60	S15VB60	S15VB60	1	SHINDENGEN	
TB	TERMINAL BOARD	10-15A	10-15A	10-15A	1	KITO	
CHM	COUNTER HOUR METER	ECP91CHAA1-3	ECP91CHAA1-3	ECP91CHAA1-3	1	OTEC	사용회수, 시간 기록
K21	MAGNET CONTACTOR	LC1K0610B7	LC1K0610B7	LC1K0610B7	1	TELEMECANIQUE	EMERGENCY STOP
K22	MAGNET CONTACTOR	HMU12	HMU12	HMU12	1	KASUGA	RIGHT/LEFT
F24	RELAY	24V	24V	24V	1	OMRON	EMERGENCY STOP
TB21	TERMINAL BOARD 21	10-15A	10-15A	10-15A	1	KITO	





접지설비 시공방법

1. 전동기의 외함, 제어반 등은 접지를 해야 하며 그 접지 저항은 다음의 규정을 따른다

접지공사	
3중 접지공사	400V 이하 100 Ω 이하
복3중 접지공사	400V 이상 10 Ω 이하

단, 방폭지역은 전압관계없이 10 Ω 이하임

2. 접지 전용 Trolley Duct 및 전선은 당해 전기기기, 기구에 대하여 충분한 용량과 전기적, 기계적인 강도를 가져야 함.

3. 접지선이 외상을 받을 우려가 있는 경우는 전선관, 합성수지관 등과 함께 사용한다

4. 접지공사는 지표면에서 최저 75cm이상의 깊이에 접지봉을 박고 접지동판에는 접지동판을 연결한다.

CABLE 종류 및 굵기

NO	FROM	CABLE SPEC.	Earth Cable
1	Main Earth Line	VCT 450x1C	User Scope(1Core)
2	Main Power Source	VCT 450 x 4C	User Scope(1Core)
3	Trolley bar or Festoon cable	60A x 4P or VCT 450 x 4C	User Scope(1Core)
4	Main Power Source	VCT 1.2550 x 6C	User Scope(1Core)
5	Traversing Motor	VCT 250 x 4C	(G-1Core)
6	Hoist Motor	VCT 1.2550 x 8C	(G-1Core)
7	Push Button Switch		(G-1Core)
8			

REV.	QTY	CONTENTS	DATE	DRAWN	APPROVED	CHECKED	DESIGNED	DRAWN	TITLE
									1.5t MOTORIZED-4점식 케이블 구성도 및 접지계통도
					M. Fukusawa	Ilinuma	A. Shimura	---	
			11.07.20	11.07.20	11.07.20				
									MDL. 942513
									DWG. NO. 3NNU942513
									SCALE NOT
									REV. 0



허용 최대 SPAN 적용표 (I-BEAM, H-BEAM)

PROJECT NAME : KD-ER2-015
 RATED LOAD : 1.5 ton
 DESCRIPTION : LIFT(max) 30 m

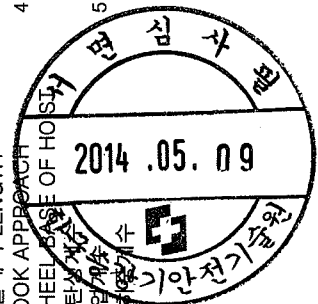
NO.	I-BEAM-SIZE (B*H*t1*t2)	Ix cm ⁴	Iy cm ⁴	Zx cm ³	Zy cm ³	A cm ²	Wb kg/m	Wh ton	Wg ton	Wg1 ton	L cm	L1 cm	L2 cm	b cm	E kg/cm ²	φ	ψ	TON/cm ²					결과			
																		Σ01	Σ02	Σ03	Σ04	δ1		L/800	δ2	L/500
I - BEAM																										
1	300x150x10/18.5t	12700	886	849	118	83.47	65.5	0.355	0.622	0.229	950	350	30	60	2100000	1.11	1.1	0.477	0.570	0.669	0.726	0.882	1.188	0.4259	0.70	O.K
2	350x150x9/15t	15200	702	870	93.5	74.58	58.5	0.355	0.585	0.228	1000	390	30	60	2100000	1.11	1.1	0.500	0.667	0.791	1.143	0.844	1.250	0.5049	0.78	O.K
3	400x150x10/18t	24100	864	1200	115	91.73	72	0.355	0.792	0.324	1100	450	30	60	2100000	1.11	1.1	0.432	0.609	0.690	1.155	0.779	1.375	0.5255	0.90	O.K
4	350x150x12/24t	22400	1180	1280	158	111.1	87.2	0.355	0.959	0.392	1100	450	30	60	2100000	1.11	1.1	0.414	0.546	0.559	0.784	0.900	1.375	0.5818	0.90	O.K
5	400x150x12.5/25t	31700	1240	1580	165	122.1	95.8	0.355	1.198	0.479	1250	500	30	60	2100000	1.11	1.1	0.429	0.532	0.677	0.880	1.025	1.563	0.5946	1.00	O.K
H - BEAM																										
6	300x150x6.5/9t	7210	508	481	67.7	40.8	32	0.355	0.24	0.09	750	280	30	60	2100000	1.11	1.1	0.556	0.729	0.777	0.898	0.626	0.938	0.3353	0.56	O.K
7	350x175x7/11t	13500	984	771	112	62.91	49.4	0.355	0.494	0.173	1000	350	30	60	2100000	1.11	1.1	0.538	0.606	0.826	0.740	0.909	1.250	0.3900	0.70	O.K
8	400x200x8/13t	23500	1740	1170	174	83.37	65.4	0.355	0.719	0.294	1100	450	30	60	2100000	1.11	1.1	0.404	0.559	0.470	0.711	0.774	1.375	0.5320	0.90	O.K
9	450x200x9/14t	32900	1870	1460	187	95.43	74.9	0.355	0.899	0.375	1200	500	30	60	2100000	1.11	1.1	0.384	0.533	0.485	0.777	0.776	1.500	0.5494	1.00	O.K

X축의 단면 2차모멘트
 Y축의 단면 2차모멘트
 Zx 축의 단면계수
 Zy 축의 단면계수
 A BEAM의 단면적
 Wb BEAM의 단위중량
 Wh HOIST 자중
 Wg GIRDER 자중
 Wg1 켈블레버 GIRDER 자중
 L SPAN-PITCH내 LENGTH
 L1 켈블레버 LENGTH
 L2 HOOK APPROACH
 b WHEEL BASE OF HOIST
 E 중탄성계수
 φ = M
 ψ = F

1. PITCH내 작업시 풍하중을 고려 허용응력에 115% 적용
 Σ01 = (PITCH내 계산응력) < 1.279 TON/CM² 이하일 경우 "O.K" (SS400, 응전효율 80% 적용, 풍하중 115% 적용)
 Σ02 = (켈블레버 계산응력) < 1.600 TON/CM² 이하일 경우 "O.K" (SS400, 풍하중 115% 적용)
 Σ03 = (PITCH내 계산응력) < 1.447 TON/CM² 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 Σ04 = (켈블레버 계산응력) < 1.808 TON/CM² 이하일 경우 "O.K" (SS400, 풍하중 130% 적용)
 1391 * 80% * 130% (휴지시 풍하중) = 1279 KG/CM²
 1391 * 115% (작업시 풍하중) = 1600 KG/CM²
 1391 * 80% * 130% (휴지시 풍하중) = 1447 KG/CM²
 1391 * 130% (휴지시 풍하중) = 1808 KG/CM²

4. < L / 800 이하일 경우 "O.K"
 < L1 / 500 이하일 경우 "O.K"

5. AREA CLASSIFICATION : IN DOOR or OUT DOOR

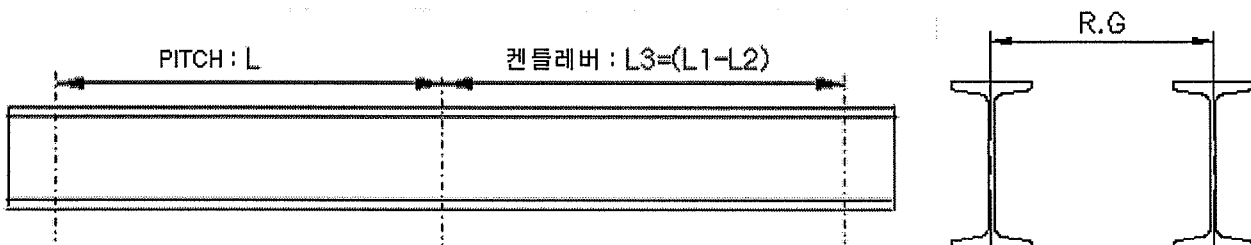


1. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.5	ton
.SPAN (PITCH)	-----	L=	950	cm
.켄틸리버	-----	L1=	350	cm
.TROLLEY WHEEL BASE	-----	B=	60	cm
.WEIGHT OF HOIST	-----	Wh=	0.355	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.622	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.229	ton
.I-BEAM SIZE	-----	300x150x10/18.5t		
		Ix =	12700	cm ⁴
		Iy =	886	cm ⁴
		Zx =	849	cm ³
		Zy =	118	cm ³
		A =	83.47	cm ²
		Wb =	65.5	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. ONE SIDE I-BEAM에 작용하는 하중

$$P = (Q + Wh)/2 = (1.5 + 0.355)/2 = 0.93 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.9275 \times (950-60/2)^2 / (4 \times 950)$$



$$= 252.2 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.9275 \times (350-30) = 362.39 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.622 \times 950 / 8 = 81.987 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.229 \times 350 / 2 = 44.48 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 334.2 \text{ ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 406.9 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 9.5 \times 0.25 \times 19.9 \times 1.7 = 89 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 3.5 \times 0.25 \times 19.9 \times 1.4 = 27 \text{ kg}$$

$$M_{FGG} = \frac{0.089 \times 950}{8} - \frac{0.027 \times 350}{2} = 5.844 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.027 \times 350}{2} = 4.725 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$HH = 1.0 \text{ m} \quad HB = 0.65 \text{ m}$$

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 950 / 4 = 4037.5 \text{ kg.cm} = 4.0375 \text{ ton.cm}$$

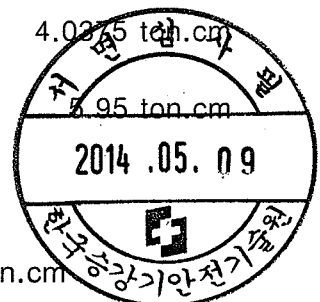
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 350 = 5950 \text{ kg.cm} = 5.95 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 5.844 + 4.0375 = 9.882 \text{ ton.cm}$$



*컨트레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 4.725 + 5.95 = 10.675 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Z_x = 334.2 / 849 = 0.394 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v2 = M_{max2} / Z_x = 406.9 / 849 = 0.479 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Z_y = 9.8815 / 118 = 0.084 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v4 = M_{HC1} / Z_y = 10.675 / 118 = 0.090 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.477 < 1.279 \text{ ton/cm}^2 \quad \text{O.K}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.570 < 1.6 \text{ ton/cm}^2 \quad \text{O.K}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
 컨트레버는 용접부 없음 $1391 \times \text{작업시} 1.15 (\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45\text{m/s}$, $q = 158.0 \text{ kg/m}^2$, $h(\text{최고양정}) = 30 \text{ m}$
 $q = M \times \sqrt{h} = 67.5 \times \sqrt{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = $F \times L \times H \times q \times 1.7 = 1.11 \times 9.5 \times 0.25 \times 158 \times 1.7 = 708 \text{ kg}$

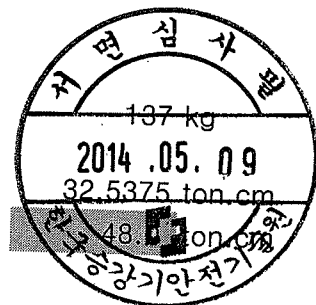
컨트레버에 대한 풍하중 = $F \times L_1 \times H \times q \times 1.4 = 1.11 \times 3.5 \times 0.25 \times 158 \times 1.4 = 215 \text{ kg}$

$$MM_{G1} = \frac{0.708 \times 950}{8} + \frac{0.215 \times 350}{2} = 46.45 \text{ ton.cm}$$

$$MM_1 = \frac{0.215 \times 350}{2} = 37.625 \text{ ton.cm}$$

HOIST에 대한 풍하중 = $F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 = 137 \text{ kg}$

*PITCH내 $MM_{H0} = 137 \times 950 / 4 = 130150 \text{ KG.CM} = 130.15 \text{ ton.cm}$
 *컨트레버 $MM_{H1} = 137 \times 350 = 47950 \text{ KG.CM} = 47.95 \text{ ton.cm}$



* COMBINED MOMENT

$$MM_2 = MM_{G1} + MM_{H0} = 46.45 + 32.5375 = 78.9875 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 37.625 + 48 = 85.63 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Zy = 78.9875 / 118 = 0.669 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Zy = 85.625 / 118 = 0.726 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 $1391 \times 80\% \times 130\% = 1447 \text{ ton/cm}^2$
 켄틀레버는 용접부 없음 휴지시 응력 $1391 \times 130\% = 1808 \text{ ton/cm}^2$

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L)^3}{48 \times E \times I_x} = 0.6212 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times Wg \times L^3}{384 \times E \times I_x} = 0.2604 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 0.882 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1078 < 800 \text{ --- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L1-L2)^3}{3 \times E \times I_x} = 0.3799 \text{ cm}$$

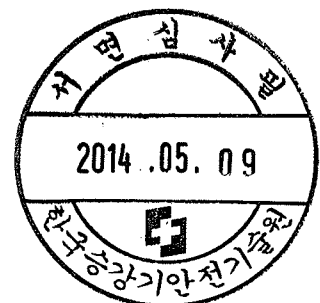
2) DUE TO DEAD LOAD

$$.D2 = \frac{Wg1 \times L1^3}{8 \times E \times I_x} = 0.0460 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 0.4259 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/822 < 500 \text{ --- O.K}$$

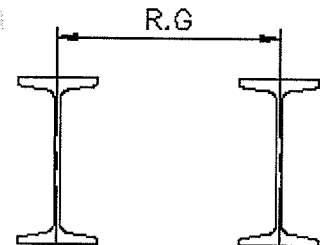
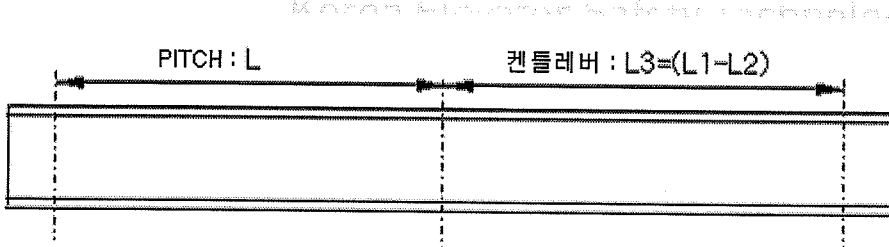


2. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.5	ton
.SPAN (PITCH)	-----	L =	1000	cm
.켄틸리버	-----	L1 =	390	cm
.TROLLEY WHEEL BASE	-----	B =	60	cm
.WEIGHT OF HOIST	-----	Wh =	0.355	ton
.PITCH내의 GIRDER 무게	-----	Wg =	0.585	ton
.켄틸리버의 GIRDER 무게	-----	Wg1 =	0.228	ton
.I-BEAM SIZE	-----		350x150x9/15t	
		Ix =	15200	cm ⁴
		Iy =	702	cm ⁴
		Zx =	870	cm ³
		Zy =	93.5	cm ³
		A =	74.58	cm ²
		Wb =	58.5	kg/m
.HOOK APPROACH	-----	L2 =	30	cm
탄성계수	-----	E =	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ) =	1.11	
정하중 계수(충격계수)	-----	F(Ψ) =	1.10	

1. DESIGN



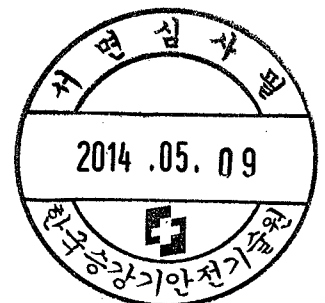
2. ONE SIDE I-BEAM에 작용하는 하중

$$P = (Q + Wh)/2 = (1.5 + 0.355)/2 = 0.93 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$\begin{aligned} Mh1 &= F \times M \times P \times (L-B/2)^2 / (4 \times L) \\ &= 1.11 \times 1.1 \times 0.9275 \times (1000-60/2)^2 / (4 \times 1000) \end{aligned}$$



$$= 266.4 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.9275 \times (390-30) = 407.69 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.585 \times 1000 / 8 = 81.169 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.228 \times 390 / 2 = 49.35 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 347.6 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 457 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 10 \times 0.25 \times 19.9 \times 1.7 = 94 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.6 = 1.11 \times 3.9 \times 0.25 \times 19.9 \times 1.6 = 34 \text{ kg}$$

$$M_{FGG} = \frac{0.094 \times 1000}{8} + \frac{0.034 \times 390}{2} = 5.12 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.034 \times 390}{2} = 6.63 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1000 / 4 = 4250 \text{ kg.cm} = 4.25 \text{ ton.cm}$$

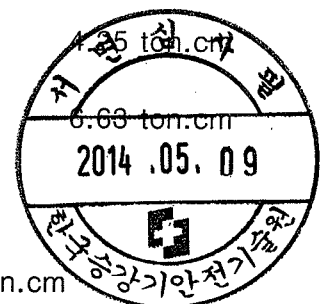
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 390 = 6630 \text{ kg.cm} = 6.63 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 5.12 + 4.25 = 9.370 \text{ ton.cm}$$



*캔틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 6.63 + 6.63 = 13.260 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Zx = 347.6 / 870 = 0.400 \text{ ton/cm}^2$$

2. 캔틀레버

$$\sigma v2 = M_{max2} / Zx = 457 / 870 = 0.525 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Zy = 9.37 / 93.5 = 0.100 \text{ ton/cm}^2$$

2. 캔틀레버

$$\sigma v4 = M_{HC1} / Zy = 13.26 / 93.5 = 0.142 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.500 < 1.279 \text{ ton/cm}^2 \quad \text{O.K}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.667 < 1.6 \text{ ton/cm}^2 \quad \text{O.K}$$

용접효율 : 80% 응력 1391 x 80% x 115% = 1.279 ton/cm²
 캔틀레버는 용접부 없음 1391 * 작업시 1.15 (풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s, q = 158.0 kg/m² h(최고양정) = 30 m
 휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²
 M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11 x 10 x 0.25 x 158 x 1.7 = 745 kg

캔틀레버에 대한 풍하중 = F x L₁ x H x q x 1.6 = 1.11 x 3.9 x 0.25 x 158 x 1.6 = 274 kg

$$MM_{G1} = \frac{0.745 \times 1000}{8} - \frac{0.274 \times 390}{2} = 39.695 \text{ ton.cm}$$

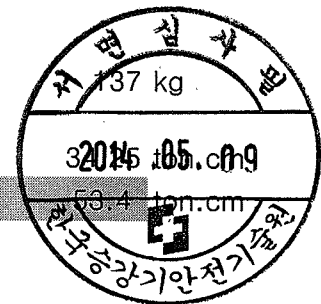
$$MM_1 = \frac{0.274 \times 390}{2} = 53.43 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x H B x H H x q x 1.2 = 1.11 x 0.65 x 1 x 158 x 1.2 =

*PITCH내 MM_{H0} = 137 x 1000 / 4 = 137000 KG.CM =

*캔틀레버 MM_{H1} = 137 x 390 = 53430 KG.CM =

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 39.695 + 34.25 = 73.945 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 53.43 + 53.4 = 106.83 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 73.945 / 93.5 = 0.791 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 106.83 / 93.5 = 1.143 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L)^3}{48 \times E \times I_x} = 0.6054 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.2386 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s1 = D1 + D2 = 0.844 \text{ cm}$$

$$\text{RATIO : } D3/L = 1 / 1185 < 800 \text{ --- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L1-L2)^3}{3 \times E \times I_x} = 0.4519 \text{ cm}$$

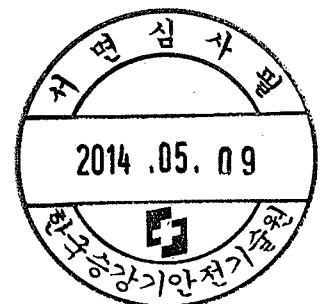
2) DUE TO DEAD LOAD

$$.D2 = \frac{W_g \times L1^3}{8 \times E \times I_x} = 0.0530 \text{ cm}$$

3) TOTAL DEFLECTION

$$.s2 = D1 + D2 = 0.5049 \text{ cm}$$

$$\text{RATIO : } D3/L = 1 / 772 < 500 \text{ --- O.K}$$

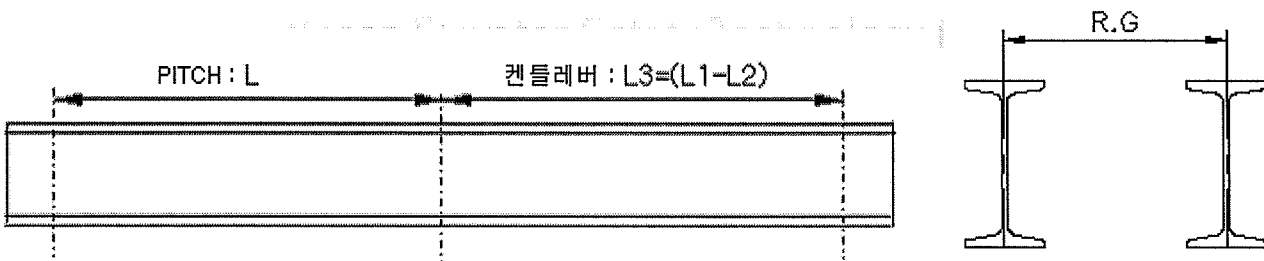


3. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.5	ton
.SPAN (PITCH)	-----	L=	1100	cm
.켄틸리버	-----	L1=	450	cm
.TROLLEY WHEEL BASE	-----	B=	60	cm
.WEIGHT OF HOIST	-----	Wh=	0.355	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.792	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.324	ton
.I-BEAM SIZE	-----	400x150x10/18t		
		Ix =	24100	cm ⁴
		Iy =	864	cm ⁴
		Zx =	1200	cm ³
		Zy =	115	cm ³
		A =	91.73	cm ²
		Wb =	72	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. ONE SIDE I-BEAM에 작용하는 하중

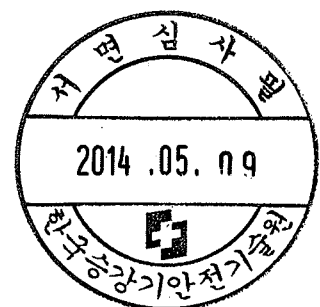
$$P = (Q + Wh)/2 = (1.5 + 0.355)/2 = 0.93 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.9275 \times (1100-60/2)^2 / (4 \times 1100)$$



$$= 294.7 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.9275 \times (450-30) = 475.64 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.792 \times 1100 / 8 = 120.879 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.324 \times 450 / 2 = 80.92 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 415.6 \text{ ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 556.6 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 11 \times 0.25 \times 19.9 \times 1.7 = 103 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.6 = 1.11 \times 4.5 \times 0.25 \times 19.9 \times 1.6 = 40 \text{ kg}$$

$$M_{FGG} = \frac{0.103 \times 1100}{8} - \frac{0.04 \times 450}{2} = 5.163 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.04 \times 450}{2} = 9 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1100 / 4 = 4675 \text{ kg.cm} =$$

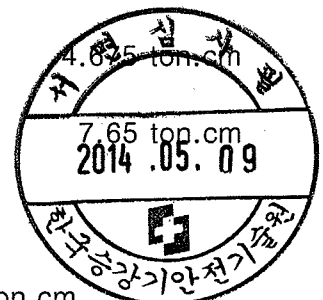
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 450 = 7650 \text{ kg.cm} =$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 5.163 + 4.675 = 9.838 \text{ ton.cm}$$



*컨트레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 9 + 7.65 = 16.650 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma_1 = M_{\max 1} / Z_x = 415.6 / 1200 = 0.346 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma_2 = M_{\max 2} / Z_x = 556.6 / 1200 = 0.464 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma_3 = M_{HCG} / Z_y = 9.838 / 115 = 0.086 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma_4 = M_{HC1} / Z_y = 16.65 / 115 = 0.145 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma_1 = \sigma_1 + \sigma_3 = 0.432 < 1.279 \text{ ton/cm}^2 \quad \text{O.K}$$

$$\Sigma \sigma_2 = \sigma_2 + \sigma_4 = 0.609 < 1.6 \text{ ton/cm}^2 \quad \text{O.K}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
 컨트레버는 용접부 없음 $1391 \times \text{작업시} 1.15 (\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45\text{m/s}$, $q = 158.0 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ m}$
 휴지시 $q = M \times \sqrt[4]{h} = 67.5 \times \sqrt[4]{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = $F \times L \times H \times q \times 1.7 = 1.11 \times 11 \times 0.25 \times 158 \times 1.7 = 820 \text{ kg}$

컨트레버에 대한 풍하중 = $F \times L_1 \times H \times q \times 1.6 = 1.11 \times 4.5 \times 0.25 \times 158 \times 1.6 = 316 \text{ kg}$

$$MM_{G1} = \frac{0.82 \times 1100}{8} - \frac{0.316 \times 450}{2} = 41.65 \text{ ton.cm}$$

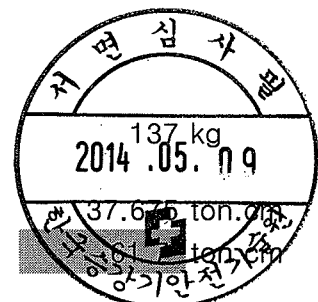
$$MM_1 = \frac{0.316 \times 450}{2} = 71.1 \text{ ton.cm}$$

HOIST에 대한 풍하중 = $F \times H_B \times H_H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$

*PITCH내 $MM_{H0} = 137 \times 1100 / 4 = 150700 \text{ KG.CM} =$

*컨트레버 $MM_{H1} = 137 \times 450 = 61650 \text{ KG.CM} =$

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 41.65 + 37.675 = 79.325 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 71.1 + 61.7 = 132.80 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Zy = 79.325 / 115 = 0.690 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Zy = 132.8 / 115 = 1.155 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 캔틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2x(L)^3}{48xEx lx} = 0.5082 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 x Wgx L^3}{384 x E x lx} = 0.2712 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d1 = D1 + D2 = 0.779 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1411 < 800 \text{ --- O.K}$$

* 캔틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2x(L1-L2)^3}{3xEx lx} = 0.4526 \text{ cm}$$

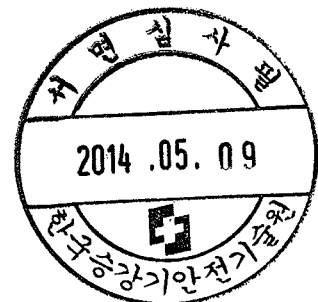
2) DUE TO DEAD LOAD

$$.D2 = \frac{Wg1x L1^3}{8 x E x lx} = 0.0729 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d2 = D1 + D2 = 0.5255 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/856 < 500 \text{ --- O.K}$$

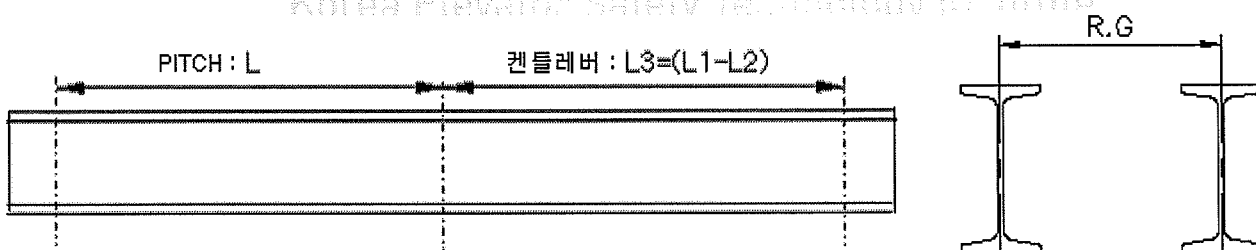


4. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.5	ton
.SPAN (PITCH)	-----	L=	1100	cm
.켄틸리버	-----	L1=	450	cm
.TROLLEY WHEEL BASE	-----	B=	60	cm
.WEIGHT OF HOIST	-----	Wh=	0.355	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.959	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.392	ton
.I-BEAM SIZE	-----		350x150x12/24t	
		Ix =	22400	cm ⁴
		Iy =	1180	cm ⁴
		Zx =	1280	cm ³
		Zy =	158	cm ³
		A =	111.1	cm ²
		Wb =	87.2	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. ONE SIDE I-BEAM에 작용하는 하중

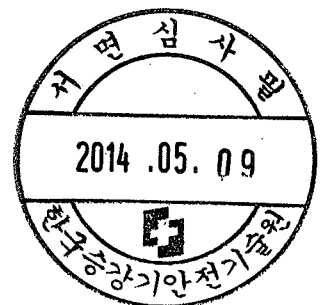
$$P = (Q + Wh)/2 = (1.5 + 0.355)/2 = 0.93 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.9275 \times (1100-60/2)^2 / (4 \times 1100)$$



$$= 294.7 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.9275 \times (450 - 30) = 475.64 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.959 \times 1100 / 8 = 146.367 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.392 \times 450 / 2 = 97.9 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 441 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 573.5 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ M}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 11 \times 0.25 \times 19.9 \times 1.7 = 103 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 4.5 \times 0.25 \times 19.9 \times 1.4 = 35 \text{ kg}$$

$$M_{FGG} = \frac{0.103 \times 1100}{8} + \frac{0.035 \times 450}{2} = 6.288 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.035 \times 450}{2} = 7.875 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1100 / 4 = 4675 \text{ kg.cm} = 4.675 \text{ ton.cm}$$

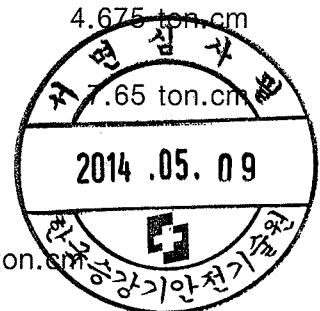
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 450 = 7650 \text{ kg.cm} = 7.65 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 6.288 + 4.675 = 10.963 \text{ ton.cm}$$



*켄틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 7.875 + 7.65 = 15.525 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma_1 = M_{max1} / Z_x = 441 / 1280 = 0.345 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma_2 = M_{max2} / Z_x = 573.5 / 1280 = 0.448 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma_3 = M_{HCG} / Z_y = 10.963 / 158 = 0.069 \text{ ton/cm}^2$$

2. 켄틀레버

$$\sigma_4 = M_{HC1} / Z_y = 15.525 / 158 = 0.098 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma_1 = \sigma_1 + \sigma_3 = 0.414 < 1.279 \text{ ton/cm}^2 \quad \text{O.K}$$

$$\Sigma \sigma_2 = \sigma_2 + \sigma_4 = 0.546 < 1.6 \text{ ton/cm}^2 \quad \text{O.K}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
 켄틀레버는 용접부 없음 $1391 \times \text{작업시} 1.15 (\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45\text{m/s}$, $q = 158.0 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ m}$
 휴지시 $q = M \times \sqrt[4]{h} = 67.5 \times \sqrt[4]{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = $F \times L \times H \times q \times 1.7 = 1.11 \times 11 \times 0.25 \times 158 \times 1.7 = 820 \text{ kg}$

켄틀레버에 대한 풍하중 = $F \times L_1 \times H \times q \times 1.4 = 1.11 \times 4.5 \times 0.25 \times 158 \times 1.4 = 276 \text{ kg}$

$$MM_{G1} = \frac{0.82 \times 1100}{8} - \frac{0.276 \times 450}{2} = 50.65 \text{ ton.cm}$$

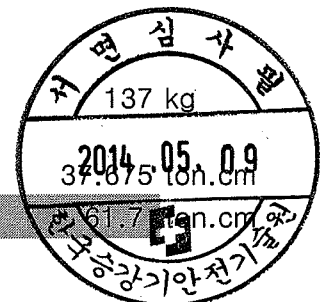
$$MM_1 = \frac{0.276 \times 450}{2} = 62.1 \text{ ton.cm}$$

HOIST에 대한 풍하중 = $F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 = 137 \text{ kg}$

*PITCH내 $MM_{H0} = 137 \times 1100 / 4 = 150700 \text{ KG.CM} = 150.7 \text{ ton.cm}$

*켄틀레버 $MM_{H1} = 137 \times 450 = 61650 \text{ KG.CM} = 61.65 \text{ ton.cm}$

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 50.65 + 37.675 = 88.325 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 62.1 + 61.7 = 123.80 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Zy = 88.325 / 158 = 0.559 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Zy = 123.8 / 158 = 0.784 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²

컨트롤레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L)^3}{48 \times E \times I_x} = 0.5467 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.3533 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d1 = D1 + D2 = 0.900 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1222 < 800 \text{ --- O.K}$$

* 컨트롤레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L1-L2)^3}{3 \times E \times I_x} = 0.4869 \text{ cm}$$

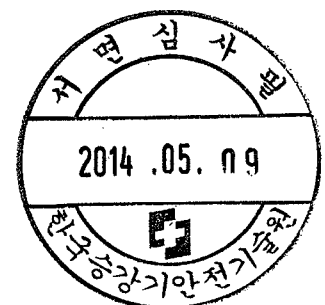
2) DUE TO DEAD LOAD

$$.D2 = \frac{W_g \times L1^3}{8 \times E \times I_x} = 0.0949 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d2 = D1 + D2 = 0.5818 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/773 < 500 \text{ --- O.K}$$

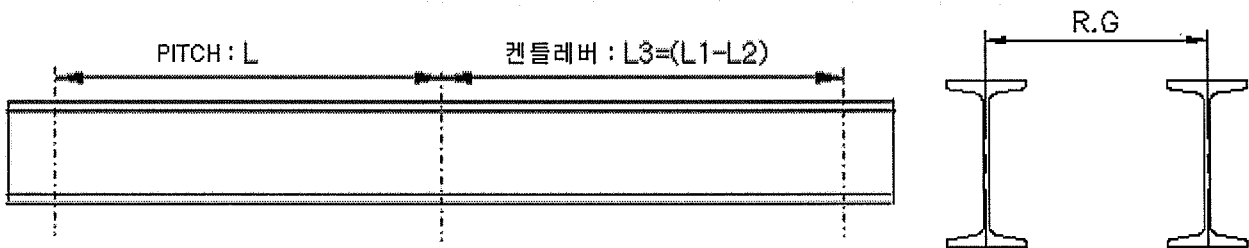


5. I-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.5	ton
.SPAN (PITCH)	-----	L=	1250	cm
.켄틸리버	-----	L1=	500	cm
.TROLLEY WHEEL BASE	-----	B=	60	cm
.WEIGHT OF HOIST	-----	Wh=	0.355	ton
.PITCH내의 GIRDER 무게	-----	Wg=	1.198	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.479	ton
.I-BEAM SIZE	-----		400x150x12.5/25t	
		Ix =	31700	cm ⁴
		Iy =	1240	cm ⁴
		Zx =	1580	cm ³
		Zy =	165	cm ³
		A =	122.1	cm ²
		Wb =	95.8	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. ONE SIDE I-BEAM에 작용하는 하중

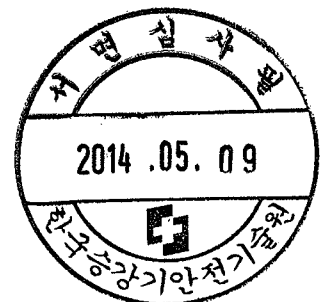
$$P = (Q + Wh)/2 = (1.5 + 0.355)/2 = 0.93 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.9275 \times (1250-60/2)^2 / (4 \times 1250)$$



$$= 337.1 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.9275 \times (500-30) = 532.26 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 1.198 \times 1250 / 8 = 207.778 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.479 \times 500 / 2 = 132.92 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 544.9 \text{ ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 665.2 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt[4]{h} = 8.5 \times \sqrt[4]{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 12.5 \times 0.25 \times 19.9 \times 1.7 = 117 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 5 \times 0.25 \times 19.9 \times 1.4 = 39 \text{ kg}$$

$$M_{FGG} = \frac{0.117 \times 1250}{8} + \frac{0.039 \times 500}{2} = 8.531 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.039 \times 500}{2} = 9.75 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1250 / 4 = 5312.5 \text{ kg.cm} = 5.3125 \text{ ton.cm}$$

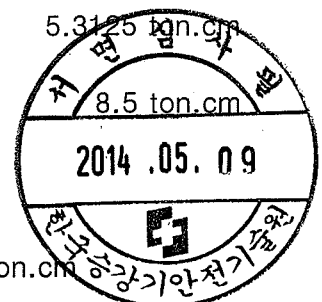
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 500 = 8500 \text{ kg.cm} = 8.5 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 8.531 + 5.3125 = 13.844 \text{ ton.cm}$$



*컨트레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 9.75 + 8.5 = 18.250 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Zx = 544.9 / 1580 = 0.345 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v2 = M_{max2} / Zx = 665.2 / 1580 = 0.421 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Zy = 13.8435 / 165 = 0.084 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v4 = M_{HC1} / Zy = 18.25 / 165 = 0.111 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.429 < 1.279 \text{ ton/cm}^2 \text{ O.K}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.532 < 1.6 \text{ ton/cm}^2 \text{ O.K}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
 컨트레버는 용접부 없음 $1391 \times \text{작업시} 1.15 (\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45\text{m/s}$, $q = 158.0 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ m}$
 $q = M \times \sqrt[4]{h} = 67.5 \times \sqrt[4]{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = $F \times L \times H \times q \times 1.7 = 1.11 \times 12.5 \times 0.25 \times 158 \times 1.7 = 932 \text{ kg}$

컨트레버에 대한 풍하중 = $F \times L_1 \times H \times q \times 1.4 = 1.11 \times 5 \times 0.25 \times 158 \times 1.4 = 307 \text{ kg}$

$$MM_{G1} = \frac{0.932 \times 1250}{8} - \frac{0.307 \times 500}{2} = 68.875 \text{ ton.cm}$$

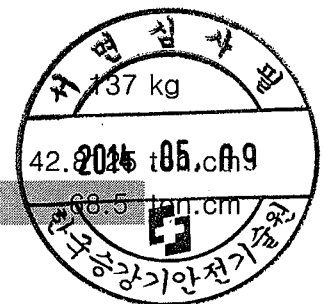
$$MM_1 = \frac{0.307 \times 500}{2} = 76.75 \text{ ton.cm}$$

HOIST에 대한 풍하중 = $F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 = 137 \text{ kg}$

*PITCH내 $MM_{H0} = 137 \times 1250 / 4 = 171250 \text{ KG.CM} = 171.25 \text{ ton.cm}$

*컨트레버 $MM_{H1} = 137 \times 500 = 68500 \text{ KG.CM} = 68.5 \text{ ton.cm}$

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 68.875 + 42.8125 = 111.6875 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 76.75 + 68.5 = 145.25 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 111.6875 / 165 = 0.677 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 145.25 / 165 = 0.880 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 캔틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L)^3}{48 \times E \times I_x} = 0.5669 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.4577 \text{ cm}$$

3) TOTAL DEFLECTION

$$. \delta_1 = D1 + D2 = 1.025 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1220 < 800 \text{ --- O.K}$$

* 캔틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L1-L2)^3}{3 \times E \times I_x} = 0.4822 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{W_{g1} \times L1^3}{8 \times E \times I_x} = 0.1124 \text{ cm}$$

3) TOTAL DEFLECTION

$$. \delta_2 = D1 + D2 = 0.5946 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/841 < 500 \text{ --- O.K}$$

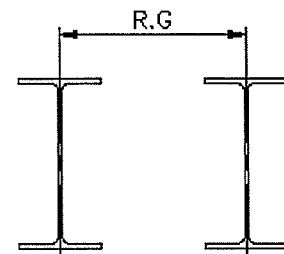
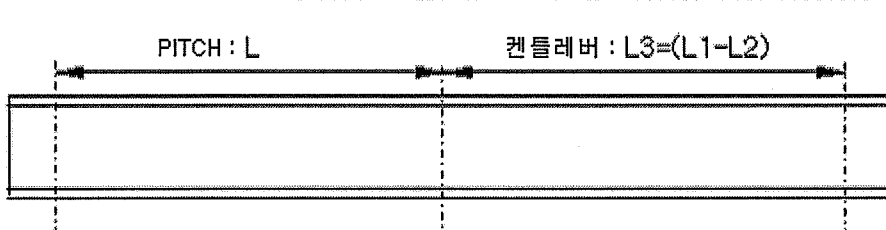


6. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.5	ton
.SPAN (PITCH)	-----	L =	750	cm
.켄틸리버	-----	L1 =	280	cm
.TROLLEY WHEEL BASE	-----	B =	60	cm
.WEIGHT OF HOIST	-----	Wh =	0.355	ton
.PITCH내의 GIRDER 무게	-----	Wg =	0.24	ton
.켄틸리버의 GIRDER 무게	-----	Wg1 =	0.09	ton
.H-BEAM SIZE	-----		300x150x6.5/9t	
		Ix =	7210	cm ⁴
		Iy =	508	cm ⁴
		Zx =	481	cm ³
		Zy =	67.7	cm ³
		A =	40.8	cm ²
		Wb =	32	kg/m
.HOOK APPROACH	-----	L2 =	30	cm
탄성계수	-----	E =	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ) =	1.11	
정하중 계수(충격계수)	-----	F(Ψ) =	1.10	

1. DESIGN



2. ONE SIDE H-BEAM에 작용하는 하중

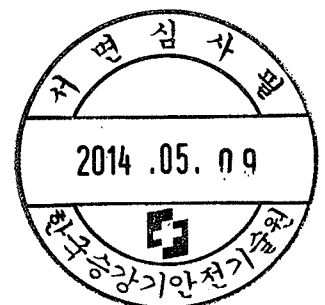
$$P = (Q + Wh)/2 = (1.5 + 0.355)/2 = 0.93 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.9275 \times (750-60/2)^2 / (4 \times 750)$$



$$= 195.7 \quad \text{ton.cm}$$

2) 캔틸라버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.9275 \times (280-30) = 283.12 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.24 \times 750 / 8 = 24.975 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.09 \times 280 / 2 = 13.99 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 220.7 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 297.1 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 7.5 \times 0.25 \times 19.9 \times 1.6 = 66 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.3 = 1.11 \times 2.8 \times 0.25 \times 19.9 \times 1.3 = 20 \text{ kg}$$

$$M_{FGG} = \frac{0.066 \times 750}{8} - \frac{0.02 \times 280}{2} = 3.388 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.02 \times 280}{2} = 2.8 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 750 / 4 = 3187.5 \text{ kg.cm} = 3.1875 \text{ ton.cm}$$

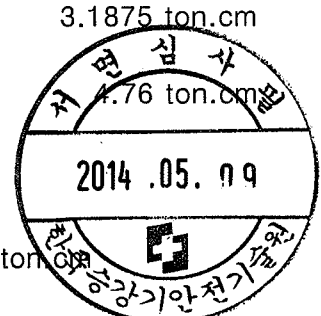
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 280 = 4760 \text{ kg.cm} = 4.76 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 3.388 + 3.1875 = 6.576 \text{ ton.cm}$$



*컨트레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 2.8 + 4.76 = 7.560 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Zx = 220.7 / 481 = 0.459 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v2 = M_{max2} / Zx = 297.1 / 481 = 0.618 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Zy = 6.5755 / 67.7 = 0.097 \text{ ton/cm}^2$$

2. 컨트레버

$$\sigma v4 = M_{HC1} / Zy = 7.56 / 67.7 = 0.112 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.556 < 1.279 \text{ ton/cm}^2 \quad \text{O.K}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.729 < 1.6 \text{ ton/cm}^2 \quad \text{O.K}$$

용접효율 : 80% 응력 1391x80% x 115% = 1.279 ton/cm²
 컨트레버는 용접부 없음 1391*작업시1.15(풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45\text{m/s}$, $q = 158.0 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ m}$
 휴지시 $q = M \times \sqrt[4]{h} = 67.5 \times \sqrt[4]{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = $F \times L \times H \times q \times 1.6 = 1.11 \times 7.5 \times 0.25 \times 158 \times 1.6 = 526 \text{ kg}$

컨트레버에 대한 풍하중 = $F \times L_1 \times H \times q \times 1.3 = 1.11 \times 2.8 \times 0.25 \times 158 \times 1.3 = 160 \text{ kg}$

$$MM_{G1} = \frac{0.526 \times 750}{8} + \frac{0.16 \times 280}{2} = 26.9125 \text{ ton.cm}$$

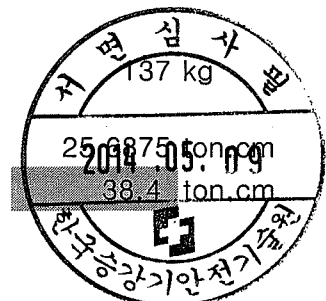
$$MM_1 = \frac{0.16 \times 280}{2} = 22.4 \text{ ton.cm}$$

HOIST에 대한 풍하중 = $F \times H_B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$

*PITCH내 $MM_{H0} = 137 \times 750 / 4 = 102750 \text{ KG.CM} =$

*컨트레버 $MM_{H1} = 137 \times 280 = 38360 \text{ KG.CM} =$

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 26.9125 + 25.6875 = 52.6 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 22.4 + 38.4 = 60.80 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 52.6 / 67.7 = 0.777 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 60.8 / 67.7 = 0.898 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% =
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% =

$$1447 \text{ ton/cm}^2$$

$$1808 \text{ ton/cm}^2$$

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L)^3}{48 \times E \times I_x} = 0.5384 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.0871 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d1 = D1 + D2 = 0.626 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1199 < 800 \text{ --- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L1-L2)^3}{3 \times E \times I_x} = 0.3190 \text{ cm}$$

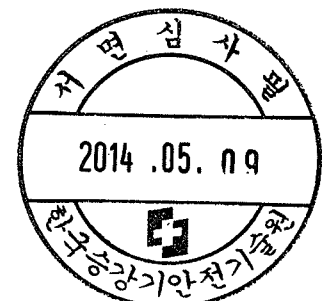
2) DUE TO DEAD LOAD

$$.D2 = \frac{W_{g1} \times L1^3}{8 \times E \times I_x} = 0.0163 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d2 = D1 + D2 = 0.3353 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/835 < 500 \text{ --- O.K}$$

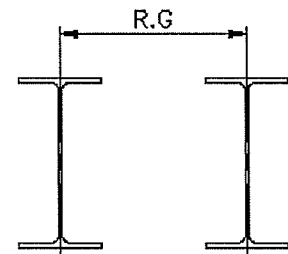
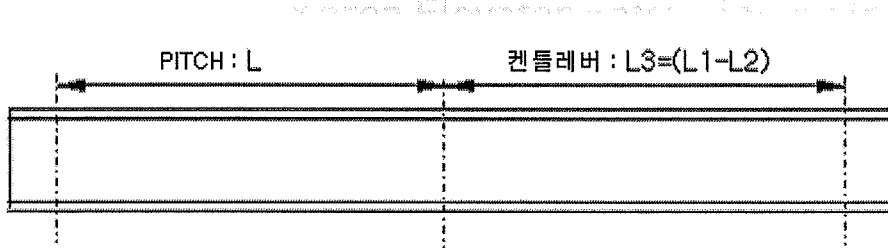


7. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.5	ton
.SPAN (PITCH)	-----	L=	1000	cm
.켄틸리버	-----	L1=	350	cm
.TROLLEY WHEEL BASE	-----	B=	60	cm
.WEIGHT OF HOIST	-----	Wh=	0.355	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.494	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.173	ton
.H-BEAM SIZE	-----		350x175x7/11t	
		Ix =	13500	cm ⁴
		Iy =	984	cm ⁴
		Zx =	771	cm ³
		Zy =	112	cm ³
		A =	62.91	cm ²
		Wb =	49.4	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. ONE SIDE H-BEAM에 작용하는 하중

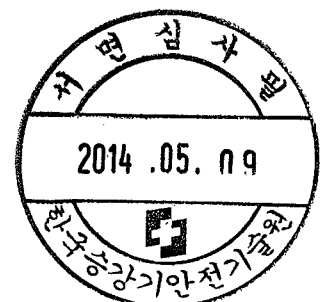
$$P = (Q + Wh)/2 = (1.5 + 0.355)/2 = 0.93 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.9275 \times (1000-60/2)^2 / (4 \times 1000)$$



$$= 266.4 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.9275 \times (350-30) = 362.39 \text{ ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.494 \times 1000 / 8 = 68.543 \text{ ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.173 \times 350 / 2 = 33.61 \text{ ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 334.9 \text{ ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 396 \text{ ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.7 = 1.11 \times 10 \times 0.25 \times 19.9 \times 1.7 = 94 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.3 = 1.11 \times 3.5 \times 0.25 \times 19.9 \times 1.3 = 25 \text{ kg}$$

$$M_{FGG} = \frac{0.094 \times 1000}{8} + \frac{0.025 \times 350}{2} = 7.375 \text{ ton.cm}$$

$$M_{FG1} = \frac{0.025 \times 350}{2} = 4.375 \text{ ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1000 / 4 = 4250 \text{ kg.cm} = 4.25 \text{ ton.cm}$$

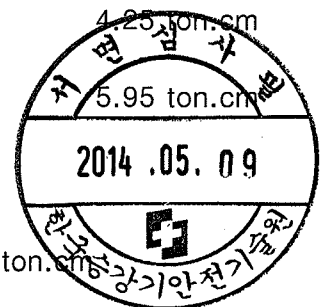
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 350 = 5950 \text{ kg.cm} = 5.95 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 7.375 + 4.25 = 11.625 \text{ ton.cm}$$



*컨틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 4.375 + 5.95 = 10.325 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Zx = 334.9 / 771 = 0.434 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma v2 = M_{max2} / Zx = 396 / 771 = 0.514 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Zy = 11.625 / 112 = 0.104 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma v4 = M_{HC1} / Zy = 10.325 / 112 = 0.092 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.538$$

0.538 < 1.279 ton/cm² O.K

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.606$$

0.606 < 1.6 ton/cm² O.K

용접효율 : 80% 응력 1391 x 80% x 115% = 1.279 ton/cm²
 컨틀레버는 용접부 없음 1391 * 작업시 1.15 (풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s, q = 158.0 kg/m² h(최고양정) = 30 m
 휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²
 M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.7 = 1.11 x 10 x 0.25 x 158 x 1.7 = 745 kg

컨틀레버에 대한 풍하중 = F x L₁ x H x q x 1.3 = 1.11 x 3.5 x 0.25 x 158 x 1.3 = 199 kg

$$MM_{G1} = \frac{0.745 \times 1000}{8} - \frac{0.199 \times 350}{2} = 58.3 \text{ ton.cm}$$

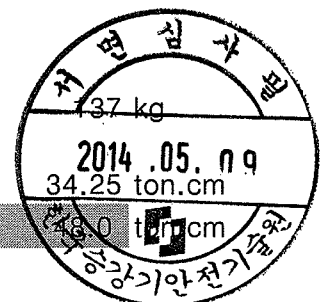
$$MM_1 = \frac{0.199 \times 350}{2} = 34.825 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x H_B x H_H x q x 1.2 = 1.11 x 0.65 x 1 x 158 x 1.2 = 137 kg

*PITCH내 MM_{H0} = 137 x 1000 / 4 = 137000 KG.CM =

*컨틀레버 MM_{H1} = 137 x 350 = 47950 KG.CM =

* COMBINED MOMENT



$$MM_2 = MM_{G1} + MM_{H0} = 58.3 + 34.25 = 92.55 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 34.825 + 48 = 82.83 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 92.55 / 112 = 0.826 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 82.825 / 112 = 0.740 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L)^3}{48 \times E \times I_x} = 0.6816 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.2269 \text{ cm}$$

3) TOTAL DEFLECTION

$$. \delta_1 = D1 + D2 = 0.909 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1101 < 800 \text{ --- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L1-L2)^3}{3 \times E \times I_x} = 0.3573 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{W_{g1} \times L^3}{8 \times E \times I_x} = 0.0327 \text{ cm}$$

3) TOTAL DEFLECTION

$$. \delta_2 = D1 + D2 = 0.3900 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/897 < 500 \text{ --- O.K}$$

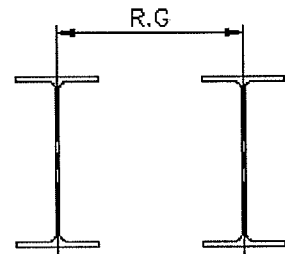
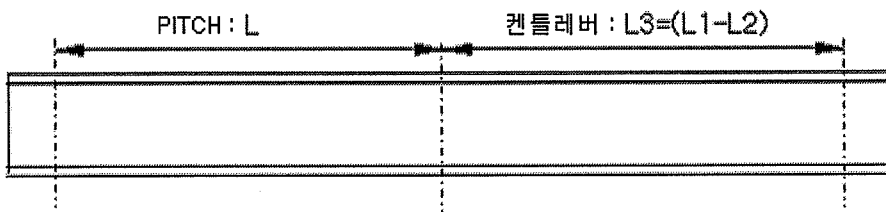


8. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.5	ton
.SPAN (PITCH)	-----	L=	1100	cm
.켄틸리버	-----	L1=	450	cm
.TROLLEY WHEEL BASE	-----	B=	60	cm
.WEIGHT OF HOIST	-----	Wh=	0.355	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.719	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.294	ton
.H-BEAM SIZE	-----		400x200x8/13t	
		Ix =	23500	cm ⁴
		Iy =	1740	cm ⁴
		Zx =	1170	cm ³
		Zy =	174	cm ³
		A =	83.37	cm ²
		Wb =	65.4	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. ONE SIDE H-BEAM에 작용하는 하중

$$P = (Q + Wh)/2 = (1.5 + 0.355)/2 = 0.93 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$Mh1 = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.9275 \times (1100-60/2)^2 / (4 \times 1100)$$



$$= 294.7 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.9275 \times (450-30) = 475.64 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.719 \times 1100 / 8 = 109.737 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.294 \times 450 / 2 = 73.43 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 404.4 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 549.1 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 11 \times 0.25 \times 19.9 \times 1.6 = 97 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 4.5 \times 0.25 \times 19.9 \times 1.4 = 35 \text{ kg}$$

$$M_{FGG} = \frac{0.097 \times 1100}{8} - \frac{0.035 \times 450}{2} = 5.463 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.035 \times 450}{2} = 7.875 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1100 / 4 = 4675 \text{ kg.cm} =$$

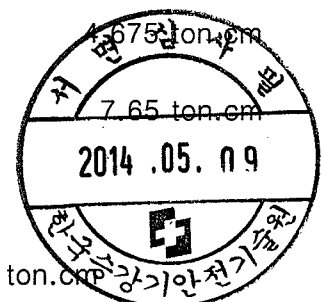
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 450 = 7650 \text{ kg.cm} =$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 5.463 + 4.675 = 10.138 \text{ ton.cm}$$



*컨틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 7.875 + 7.65 = 15.525 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Zx = 404.4 / 1170 = 0.346 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma v2 = M_{max2} / Zx = 549.1 / 1170 = 0.469 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Zy = 10.138 / 174 = 0.058 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma v4 = M_{HC1} / Zy = 15.525 / 174 = 0.089 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.404 < 1.279 \text{ ton/cm}^2 \quad \text{O.K}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.559 < 1.6 \text{ ton/cm}^2 \quad \text{O.K}$$

용접효율 : 80% 응력 $1391 \times 80\% \times 115\% = 1.279 \text{ ton/cm}^2$
 컨틀레버는 용접부 없음 $1391 * \text{작업시} 1.15 (\text{풍하중포함}) = 1.600 \text{ ton/cm}^2$

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 $V=45\text{m/s}$, $q = 158.0 \text{ kg/m}^2$ $h(\text{최고양정}) = 30 \text{ m}$
 휴지시 $q = M \times \sqrt[4]{h} = 67.5 \times \sqrt[4]{30} = 158.0 \text{ kg/m}^2$
 $M = V^2 / 30 = 67.5$

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = $F \times L \times H \times q \times 1.6 = 1.11 \times 11 \times 0.25 \times 158 \times 1.6 = 772 \text{ kg}$

컨틀레버에 대한 풍하중 = $F \times L_1 \times H \times q \times 1.4 = 1.11 \times 4.5 \times 0.25 \times 158 \times 1.4 = 276 \text{ kg}$

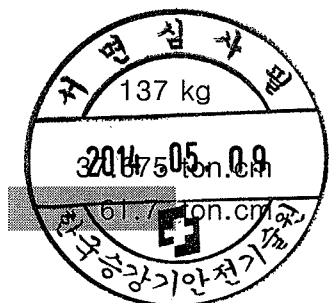
$$MM_{G1} = \frac{0.772 \times 1100}{8} - \frac{0.276 \times 450}{2} = 44.05 \text{ ton.cm}$$

$$MM_1 = \frac{0.276 \times 450}{2} = 62.1 \text{ ton.cm}$$

HOIST에 대한 풍하중 = $F \times H \times B \times H \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 158 \times 1.2 =$

*PITCH내 $MM_{H0} = 137 \times 1100 / 4 = 150700 \text{ KG.CM} = 150.7 \text{ ton.cm}$

*컨틀레버 $MM_{H1} = 137 \times 450 = 61650 \text{ KG.CM} = 61.65 \text{ ton.cm}$



* COMBINED MOMENT

$$MM_2 = MM_{G1} + MM_{H0} = 44.05 + 37.675 = 81.725 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 62.1 + 61.7 = 123.80 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Zy = 81.725 / 174 = 0.470 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Zy = 123.8 / 174 = 0.711 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 $1391 \times 80\% \times 130\% = 1447 \text{ ton/cm}^2$
 캔틀레버는 용접부 없음 휴지시 응력 $1391 \times 130\% = 1808 \text{ ton/cm}^2$

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L)^3}{48 \times E \times I_x} = 0.5212 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.2525 \text{ cm}$$

3) TOTAL DEFLECTION

$$. \delta_1 = D1 + D2 = 0.774 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1422 < 800 \text{ --- O.K}$$

* 캔틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L1-L2)^3}{3 \times E \times I_x} = 0.4641 \text{ cm}$$

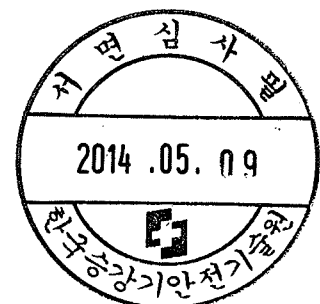
2) DUE TO DEAD LOAD

$$.D2 = \frac{W_g \times L1^3}{8 \times E \times I_x} = 0.0679 \text{ cm}$$

3) TOTAL DEFLECTION

$$. \delta_2 = D1 + D2 = 0.5320 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/846 < 500 \text{ --- O.K}$$

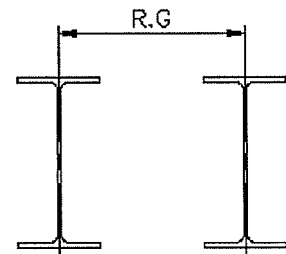
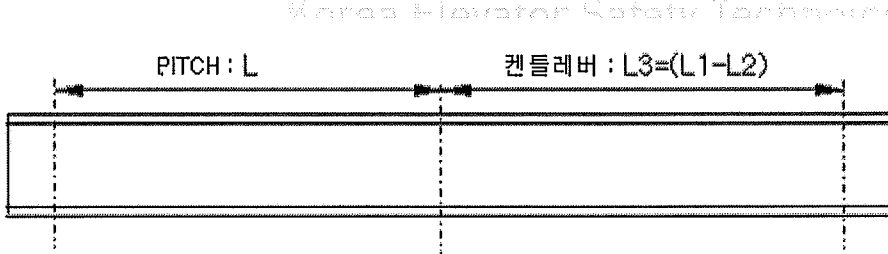


9. H-BEAM 최대허용 가능 SPAN 계산

1. SPECIFICATION

.정격하중	-----	Q =	1.5	ton
.SPAN (PITCH)	-----	L=	1200	cm
.켄틸리버	-----	L1=	500	cm
.TROLLEY WHEEL BASE	-----	B=	60	cm
.WEIGHT OF HOIST	-----	Wh=	0.355	ton
.PITCH내의 GIRDER 무게	-----	Wg=	0.899	ton
.켄틸리버의 GIRDER 무게	-----	Wg1=	0.375	ton
.H-BEAM SIZE	-----		450x200x9/14t	
		Ix =	32900	cm ⁴
		Iy =	1870	cm ⁴
		Zx =	1460	cm ³
		Zy =	187	cm ³
		A =	95.43	cm ²
		Wb =	74.9	kg/m
.HOOK APPROACH	-----	L2=	30	cm
탄성계수	-----	E=	2100000	kg/cm ²
동하중 계수(작업계수)	-----	M(Φ)=	1.11	
정하중 계수(충격계수)	-----	F(Ψ)=	1.10	

1. DESIGN



2. ONE SIDE H-BEAM에 작용하는 하중

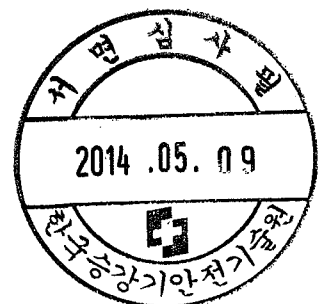
$$P = (Q + Wh)/2 = (1.5 + 0.355)/2 = 0.93 \text{ ton}$$

3. 수직하중에 의한 BENDING MOMENT

1) PITCH 지점내 BENDING MOMENT

$$M_{h1} = F \times M \times P \times (L-B/2)^2 / (4 \times L)$$

$$= 1.11 \times 1.1 \times 0.9275 \times (1200-60/2)^2 / (4 \times 1200)$$



$$= 323.0 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mh2 = F \times M \times P \times L3$$

$$= 1.11 \times 1.1 \times 0.9275 \times (500-30) = 532.26 \quad \text{ton.cm}$$

4. 자중에 의한 BENDING MOMENT(I-BEAM)

1) PITCH 지점내 BENDING MOMENT

$$Mt1 = F \times Wg \times L / 8 = 1.11 \times 0.899 \times 1200 / 8 = 149.684 \quad \text{ton.cm}$$

2) 캔틸리버 BENDING MOMENT

$$Mt2 = F \times Wg1 \times L1 / 2 = 1.11 \times 0.375 \times 500 / 2 = 104.06 \quad \text{ton.cm}$$

5. 수직하중에 의한 최대 BENDING MOMENT

$$Mmax1 = Mh1 + Mt1 = 472.7 \quad \text{ton.cm}$$

$$Mmax2 = Mh2 + Mt2 = 636.3 \quad \text{ton.cm}$$

6. 풍하중에 의한 수평하중의 BENDING MOMENT

(1) GIRDER에 의한 수직 풍하중

$$\text{작업시 } V=16\text{m/s}, q = 19.9 \text{ kg/m}^2 \quad h(\text{최고양정}) = 30 \text{ m}$$

$$\text{작업시 } q = 8.5 \times \sqrt{h} = 8.5 \times \sqrt{42} = 19.9 \text{ kg/m}^2$$

$$H(\text{GIRDER높이}) = 0.25 \text{ m}$$

$$\text{PITCH내 풍하중} = F \times L \times H \times q \times 1.6 = 1.11 \times 12 \times 0.25 \times 19.9 \times 1.6 = 106 \text{ kg}$$

$$\text{캔틀레버 풍하중} = F \times L1 \times H \times q \times 1.4 = 1.11 \times 5 \times 0.25 \times 19.9 \times 1.4 = 39 \text{ kg}$$

$$M_{FGG} = \frac{0.106 \times 1200}{8} - \frac{0.039 \times 500}{2} = 6.15 \quad \text{ton.cm}$$

$$M_{FG1} = \frac{0.039 \times 500}{2} = 9.75 \quad \text{ton.cm}$$

(2) HOIST에 의한 풍하중

* 작업시

$$\text{풍하중} = F \times HB \times HH \times q \times 1.2 = 1.11 \times 0.65 \times 1 \times 19.9 \times 1.2 = 17 \text{ kg}$$

*PITCH내 풍하중

$$M_{FHG} = 17 \times 1200 / 4 = 5100 \text{ kg.cm} = 5.1 \text{ ton.cm}$$

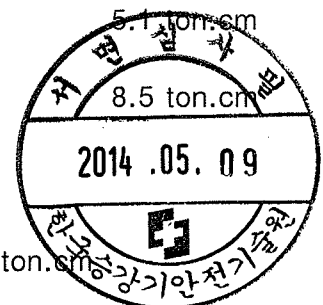
*캔틀레버 풍하중

$$M_{FH1} = 17 \times 500 = 8500 \text{ kg.cm} = 8.5 \text{ ton.cm}$$

7. COMBINED MOMENT

*PITCH내

$$M_{HCG} = M_{FGG} + M_{FHG} = 6.15 + 5.1 = 11.250 \text{ ton.cm}$$



*컨틀레버

$$M_{HC1} = M_{FG1} + M_{FH1} = 9.75 + 8.5 = 18.250 \text{ ton.cm}$$

8. BENDING STRESS

A. VERTICAL BENDING STRESS

1. PITCH 내

$$\sigma v1 = M_{max1} / Zx = 472.7 / 1460 = 0.324 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma v2 = M_{max2} / Zx = 636.3 / 1460 = 0.436 \text{ ton/cm}^2$$

B. 수평 최대 응력

1. PITCH 내

$$\sigma v3 = M_{HCG} / Zy = 11.25 / 187 = 0.060 \text{ ton/cm}^2$$

2. 컨틀레버

$$\sigma v4 = M_{HC1} / Zy = 18.25 / 187 = 0.098 \text{ ton/cm}^2$$

9. 합성 응력

$$\Sigma \sigma 1 = \sigma v1 + \sigma v3 = 0.384 < 1.279 \text{ ton/cm}^2 \quad \text{O.K}$$

$$\Sigma \sigma 2 = \sigma v2 + \sigma v4 = 0.533 < 1.6 \text{ ton/cm}^2 \quad \text{O.K}$$

용접효율 : 80% 응력 1391 x 80% x 115% = 1.279 ton/cm²
 컨틀레버는 용접부 없음 1391 * 작업시 1.15 (풍하중포함) = 1.600 ton/cm²

10. 휴지시 풍하중에 의한 BENDING MOMENT

휴지시 V=45m/s, q = 158.0 kg/m² h(최고양정) = 30 m
 휴지시 q = M x ⁴√h = 67.5 x ⁴√30 = 158.0 kg/m²
 M = V² / 30 = 67.5

*휴지시 풍하중에 의한 수평하중

PITCH내 풍하중 = F x L x H x q x 1.6 = 1.11 x 12 x 0.25 x 158 x 1.6 = 842 kg

컨틀레버에 대한 풍하중 = F x L x H x q x 1.4 = 1.11 x 5 x 0.25 x 158 x 1.4 = 307 kg

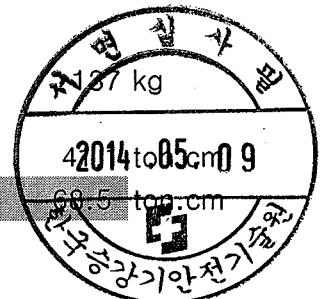
$$MM_{G1} = \frac{0.842 \times 1200}{8} - \frac{0.307 \times 500}{2} = 49.55 \text{ ton.cm}$$

$$MM_1 = \frac{0.307 \times 500}{2} = 76.75 \text{ ton.cm}$$

HOIST에 대한 풍하중 = F x HB x HH x q x 1.2 = 1.11 x 0.65 x 1 x 158 x 1.2 =

*PITCH내 MM_{H0} = 137 x 1200 / 4 = 164400 KG.CM =

*컨틀레버 MM_{H1} = 137 x 500 = 68500 KG.CM =



* COMBINED MOMENT

$$MM_2 = MM_{G1} + MM_{H0} = 49.55 + 41.1 = 90.65 \text{ ton.cm}$$

$$MM_4 = MM_1 + MM_{H1} = 76.75 + 68.5 = 145.25 \text{ ton.cm}$$

* BENDING STRESS

$$\Sigma\sigma_3 = MM_2 / Z_y = 90.65 / 187 = 0.485 \text{ ton/cm}^2 < 1.447 \text{ ton/cm}^2 \text{---O.K}$$

$$\Sigma\sigma_4 = MM_4 / Z_y = 145.25 / 187 = 0.777 \text{ ton/cm}^2 < 1.808 \text{ ton/cm}^2 \text{---O.K}$$

PITCH내 휴지시 응력 1391 x 80% x 130% = 1447 ton/cm²
 켄틀레버는 용접부 없음 휴지시 응력 1391 x 130% = 1808 ton/cm²

11. DEFLECTION OF GIRDER

* PITCH 내

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L)^3}{48 \times E \times I_x} = 0.4833 \text{ cm}$$

2) DUE TO DEAD LOAD

$$.D2 = \frac{5 \times W_g \times L^3}{384 \times E \times I_x} = 0.2928 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d1 = D1 + D2 = 0.776 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/1546 < 800 \text{ --- O.K}$$

* 켄틀레버

1) DUE TO RATED & TROLLEY LOAD

$$.D1 = \frac{(Q+Q2)/2 \times (L1-L2)^3}{3 \times E \times I_x} = 0.4646 \text{ cm}$$

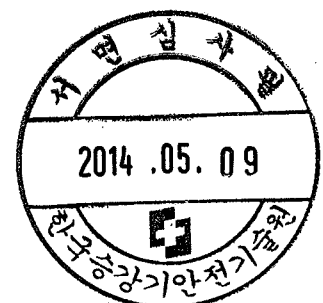
2) DUE TO DEAD LOAD

$$.D2 = \frac{W_g \times L1^3}{8 \times E \times I_x} = 0.0848 \text{ cm}$$

3) TOTAL DEFLECTION

$$.d2 = D1 + D2 = 0.5494 \text{ cm}$$

$$\text{RATIO : } D3/L = 1/910 < 500 \text{ --- O.K}$$



SUSPENSION MONORAIL 볼트 및 용접강도계산

1. 볼트로 고정시의 강도계산

* 허용 최대 SPAN 및 하중은 최악의 조건으로 계산한다.
(허용 최대 스팬 12.5 M, 400x150x12.5/25t, HOIST자중 : 355 KG)

.Q = RATED LOAD= 1500 KG
.Q1=HOIST SELF WEIGHT= 355 KG
.Q2=I-BEAM WEIGHT = 1198 KG

BOLT 재질	H.T.B 9.8 이상 사용할것
BOLT 허용전단응력	2100 KG/CM ²

M14 일때 do= 1.1835 CM n= 4.4 . n = 유효산수(산수*80%너트1종너트사용)
M16 일때 do= 1.3835 CM n= 5.2
M18 일때 do= 1.5294 CM n= 4.8
M20 일때 do= 1.7294 CM n= 5.1

$$P = 1.14*(Q+Q1)/2 + 1.1* Q2/2$$

$$= 1.14*(1500+355)/2 + 1.1* 1198/2$$

$$= 1716 \text{ KG} \quad (\sigma_a = 2100 \text{ KG/CM}^2)$$

1)인장(전단)강도 : $\sigma = P/A$; $\sigma = \frac{4 \times P}{\pi \times do^2 \times Z}$ * $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma}$

M16 일때 $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 1716}{835^2 \times 2100} = 0.54 \text{ 개}$

M18 일때 $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 1716}{294^2 \times 2100} = 0.44 \text{ 개}$

M20 일때 $Z = \frac{4 \times P}{\pi \times do^2 \times \sigma} = \frac{4 \times 1716}{294^2 \times 2100} = 0.35 \text{ 개}$

2)접촉 면압 강도 ($\sigma_a = 400 \text{ KG/CM}^2$)

$$\sigma = \frac{4 \times P}{\pi \times (d^2 - do^2) \times Z \times n} ; \quad Z = \frac{4 \times P}{\pi \times (d^2 - do^2) \times \sigma \times n}$$

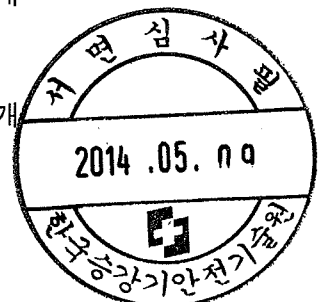
M14 일때 $Z = \frac{4 \times 1716}{\pi \times (1.4^2 - 1.1835^2) \times 400 \times 4.4} = 2.22 \text{ 개}$

M16 일때 $Z = \frac{4 \times 1716}{\pi \times (1.6^2 - 1.3835^2) \times 400 \times 5.2} = 1.63 \text{ 개}$

M18 일때 $Z = \frac{4 \times 1716}{\pi \times (1.8^2 - 1.5294^2) \times 400 \times 4.8} = 1.26 \text{ 개}$

M20 일때 $Z = \frac{4 \times 1716}{\pi \times (2^2 - 1.7294^2) \times 400 \times 5.1} = 1.06 \text{ 개}$

. do = 골경, P=브라켓 한 개에 작용하는 하중, Z = 볼트수
. n = 유효산수(산수*80%), 너트1종너트사용



3) 따라서 다음과 같이 적용한다

H.T.B M14, M16, M18, M20일때 : 브라켓트당 4개 이상 사용한다.

2. I-빔을 용접시공으로 고정시의 강도계산

* 허용 최대 SPAN 및 하중은 최악의 조건으로 계산한다.

(허용 최대 스팬 12.5 M, 400x150x12.5/25t, HOIST자중 : 355 KG)

.Q = RATED LOAD = 1500 KG (h : 용접두께)
 .Q1=HOIST SELF WEIGHT = 355 KG (L : 용접길이)
 .Q2=I-BEAM WEIGHT = 1198 KG

1) 용접이음부 인장강도 계산 (용접두께 : 45° 용접부위)

$$\sigma = \frac{1.414 \times P}{h \times L}; (\sigma_a = 1200 \text{ KG/CM}^2) \quad L = \frac{1.414 \times P}{h \times \sigma}$$

(1) h 가 5일때

$$L = \frac{1.414 \times 1716}{0.5 \times 1200} = 4.04 \text{ CM} = 40.4 \text{ mm}$$

(2) h 가 6일때

$$L = \frac{1.414 \times 1716}{0.6 \times 1200} = 3.37 \text{ CM} = 33.7 \text{ mm}$$

(3) h 가 7일때

$$L = \frac{1.414 \times 1716}{0.7 \times 1200} = 2.89 \text{ CM} = 28.9 \text{ mm}$$

2) 적용

h = 5일때, 한 브라켓트당 용접길이 L = 60mm 이상 용접한다.

h = 6일때, 한 브라켓트당 용접길이 L = 50mm 이상 용접한다.

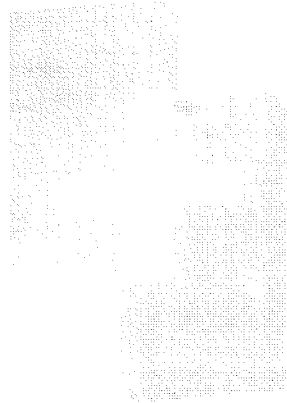
h = 7일때, 한 브라켓트당 용접길이 L = 40mm 이상 용접한다.

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6. FOR REFERENCE

- 1) LOAD CHAIN 시험성적서
- 2) MOTOR DATA SHEET
- 3) HOIST 사용설명서(operation manual)



한국승강기안전기술원
Korea Elevator Safety Technology Institute

Date: 2009/04/14

Certificate of Compliance

We certify that the ER2 protection degrees conform to the IP rating as follows:

Hoist body - IP55 based on JIS C 4034-5, "Rotating electrical machines – Part5: Classification of degrees of protection provided by enclosures of rotating electrical machines (IP code)".

Push button - IP65 based on JIS C 0920, "Tests to prove protection against ingress of water and degrees of protection against ingress of solid objects for electrical equipment".

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Technical Control Group

Test Certificate

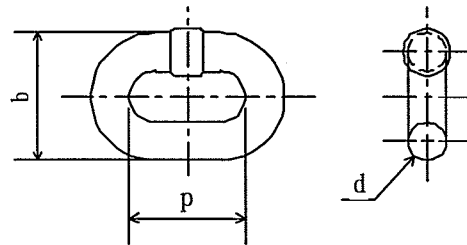
Messrs.

Commodity: NC Load Chain

Code: KER102

Lot No.: —

Quantity: — line(s)



1. Material: Manganese Alloy Steel

2. Dimensions

	d	p	b
Specified	10.2mm ± 0.4	28.4mm $\begin{matrix} +0.56 \\ 0 \end{matrix}$	Max. 35.7mm
Result	Good	Good	Good

3. Breaking test

	Breaking load	Total ultimate elongation
Specified	Min. 131 (kN)	Min. 10 (%)
Result	Good	Good

4. Manufacturing Proof force test (Test load: 81.7 kN)

	Permanent elongation
Specified	0.25 (%)
Result	Good

General judgment: Satisfactory



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Quality Assurance Group
Quality Assurance Department
Development & Technology Division

R. Kishimoto (Manager)

Certificate No.: MM080011e

Date of Issue: 2009/3/4

Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	1.8kW	4P	60%ED	220(208)V	60Hz

Full load characteristics

Voltage Frequency	220V 60Hz	
Load %	100	
Current A	8.4	
Speed rpm	1620	

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Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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Quality Assurance Department
Development & Technology Division

M. Ogihara

(Manager)

Certificate No.: MM070011e

Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	1.8kW	4P	60%ED	380 - 440V	60Hz

Full load characteristics

Voltage	Frequency	380 - 440V	60Hz
Load	%	100	
Current	A	4.6	
Speed	rpm	1610	

Korea Elevator Safety Technology Institute

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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Quality Assurance Department
Development & Technology Division

(Manager)

K. Kishimoto

Certificate No.: MM080011f

Date of Issue: 2009/3/4

Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	1.8kW	4P	40/20%ED	220(208)V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	220V	Speed Control by Inverter
Load	%		100
Current	A		11.2
Speed	rpm		~

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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Development & Technology Division

M. Ogihara

(Manager)

Certificate No.: MM070011f

Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for Electric Chain Hoist

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ	1.8kW	4P	60%ED	380 - 440V	Speed Control by Inverter

Full load characteristics

Voltage Frequency	380 - 440V	Speed Control by Inverter
Load %		100
Current A		5.1
Speed rpm		~

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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Date of Issue: 2009/3/4

Messrs. _____

Motor Test Report for Electric Trolley

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	220(208)V	60Hz

Full load characteristics

Voltage Frequency	220V 60Hz
Load %	100
Current A	3.0
Speed rpm	1685

Korea Elevator Safety Technology Institute

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric trolley and the trolley is subjected to full load



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Development & Technology Division

M. Ogihara

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Certificate No.: MM070013a

Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for End Carriage

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	380 - 440V	60Hz

Full load characteristics

Voltage Frequency	380 - 440V 60Hz	
Load	%	100
Current	A	2.2
Speed	rpm	1670

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Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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(Manager)

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Date of Issue: 2009/3/4

Messrs. _____

Motor Test Report for Electric Trolley

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. : -

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	27/13%ED	220(208)V	Speed Control by Inverter

Full load characteristics

Voltage	Frequency	220V	Speed Control by Inverter
Load	%		100
Current	A		3.0
Speed	rpm		~

Korea Elevator Safety Technology Institute

Insulation class E

The above characteristics are obtained from calculation where the motor is assembled with an electric trolley and the trolley is subjected to full load



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Certificate No.: MM070013b

Date of Issue: 2008/03/21

Messrs. _____

Motor Test Report for End Carriage

Motor type : Three phase squirrel cage type induction motor.

Manufacturer : Yasukawa Electric Mfg. Co.

Production No. :

Rating

Model	Output	Pole	Intermittent Rating	Voltage	Frequency
IBQ-T	0.4kW	4P	40%ED	380 - 440V	Speed Control by Inverter

Full load characteristics

Voltage Frequency	220 - 230V	Speed Control by Inverter
Load %	100	
Current A	2.5	
Speed rpm	~	

Insulation class B

The above characteristics are obtained from calculation where the motor is assembled with an electric chain hoist and the hoist is subjected to full load



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